



**U.S. MAGNET  
DEVELOPMENT  
PROGRAM**



# US Magnet Development Program

## Design of a shell-based Utility Structure

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- **Goals of the utility structure design effort**
- **Shell based structure concept**
- **Present and future coil parameters**
- **Utility structure cross-section details**
- **2D and 3D model results for 15T coil pre-load**
- **Towards 16 T coil**
- **Conclusions**

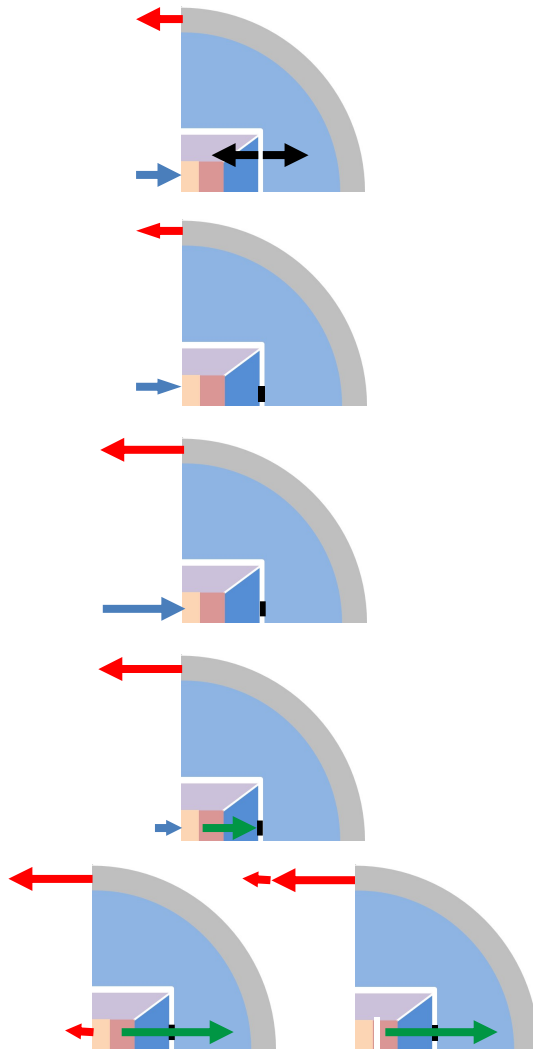


## Developing a shell-based magnet structure to be used for testing MDP coils

- **Provide adequate pre-stress for 17T operation**
  - 200MPa peak compressive stress
  - Prevent tensile stress in the pole area (pole-turn separation)
  - Rapid and reproducible magnet assembly/disassembly
  - Compatible with the existing 15 T 4-layer Cos-theta magnet design (FNAL)
- **Investigation of design limits and sensitivities**
  - Close coil-structure design effort
  - Impact of structure dimensions and features on pre-load capability
  - Coil features and fabrication technology impact on the ability to provide pre-load
  - Effect of fabrication tolerances on mechanical performance



# Bladder and Key pre-load concept in a Shell-based structure



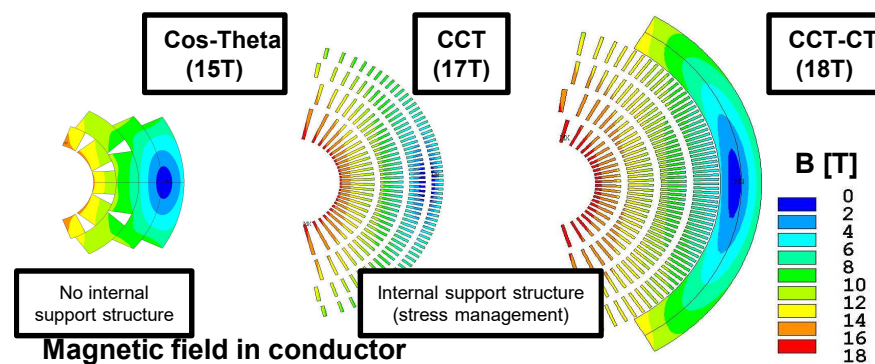
- **Bladder operation**
  - Pressure acting on the yoke side compensated by reaction force in the shell (tension)
  - Pressure acting on the pad side compensated by the reaction force in the pole (compression)
- **Room temperature pre-load**
  - Keys replace bladders
  - Reaction forces drop slightly (~10-20%)
- **Cool-down**
  - Reaction forces increase due to aluminum shell shrinkage
- **Magnetic forces (coil pre-loaded)**
  - Reaction force in the shell remains almost the same as after cool-down
  - Reaction force in the pole decreases due to magnetic force
- **Magnetic forces (coil un-loaded)**
  - Magnetic forces higher than pre-load
  - Coil can separate from the pole
  - Reaction force in the pole drops to zero
  - Reaction force in the shell increases due to magnetic forces



# Coil parameters

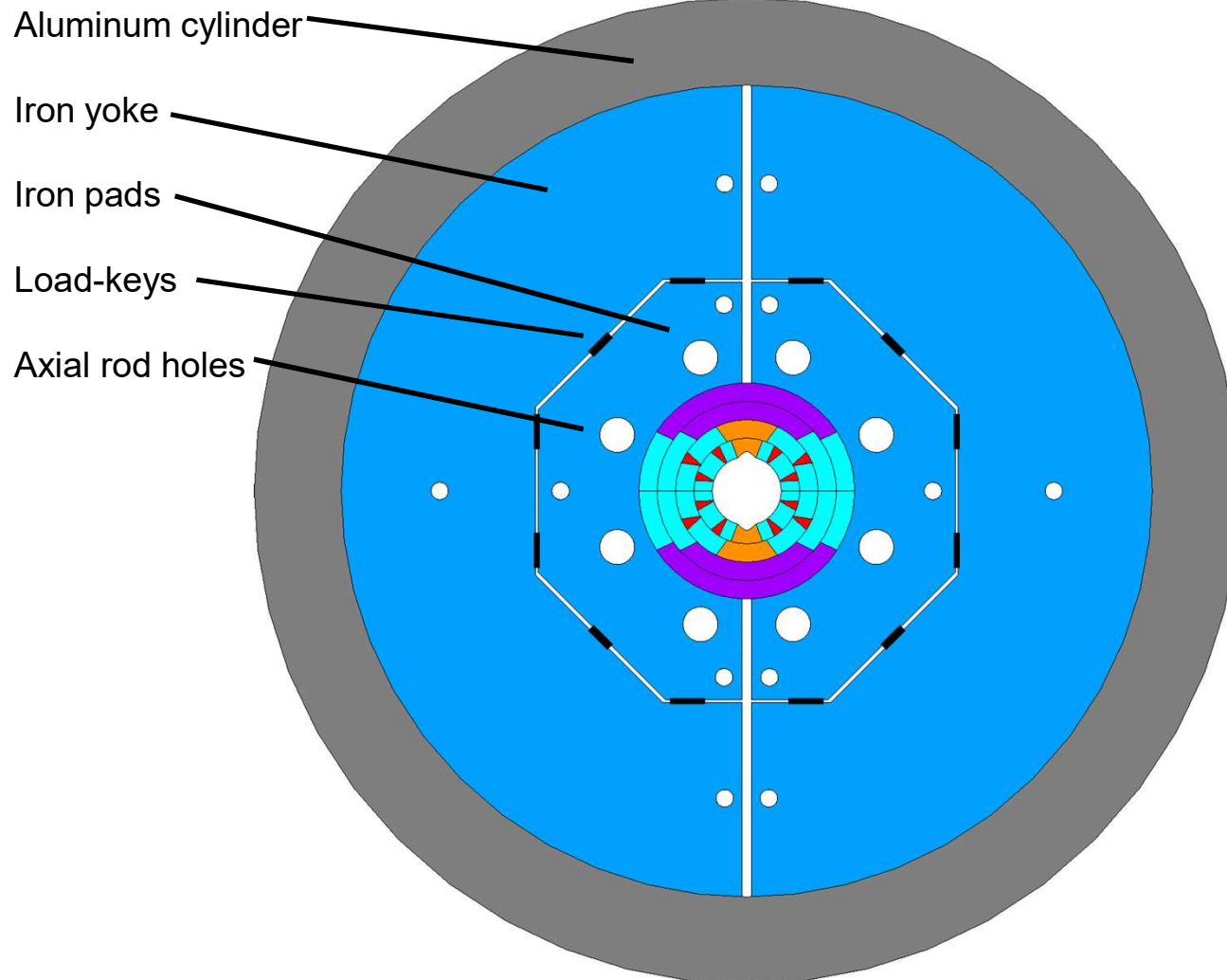
[preliminary/conceptual/approximate]

	B [T]	$F_{x_{quad}}$ [MN/m]	$F_{y_{quad}}$ [MN/m]	OD [mm]
CT	15 15.6	6.8 7.4	-3.9 -4.5	188
CT-SM	16	9.5	4.1	208
CCT	17	11		194
CCT-CT	18	14		256
HTS/LTS	>16	?	?	< 280 ?





# Structure with octagonal coil-pack

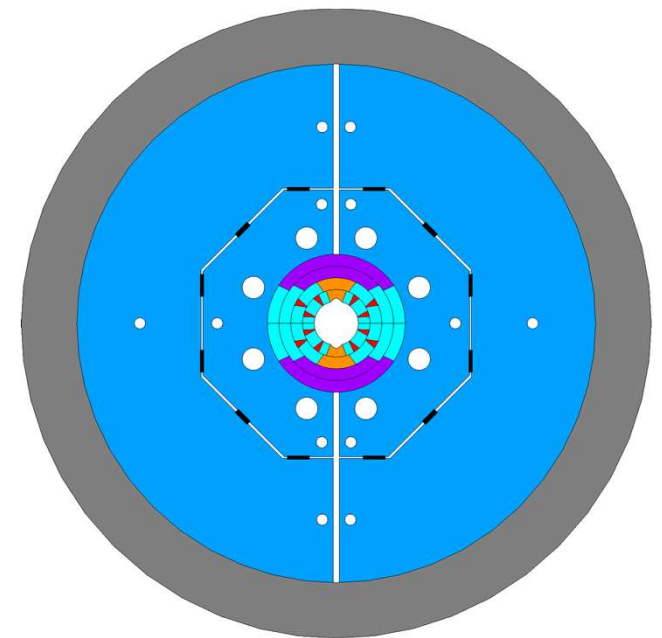


- **Structure with octagonal coil-pack**
  - Improves yoke stress distribution and rigidity
  - Coil-pack horizontal and vertical size = 320 mm
    - Smaller coil – axial rods in the coil-pack
    - Bigger coils – axial rods in the yoke
- **Three load-keys**
  - Horizontal
    - Pre-load function
  - Diagonal
    - Stress distribution and rigidity
  - Vertical
    - Alignment
  - Possibility of closing yoke gap
- **Bladders**
  - Mid-plane (2)
  - Diagonal (8)
  - Yoke (4 or 6)
  - Vertical (2)



# Structure and coil-pack dimensions

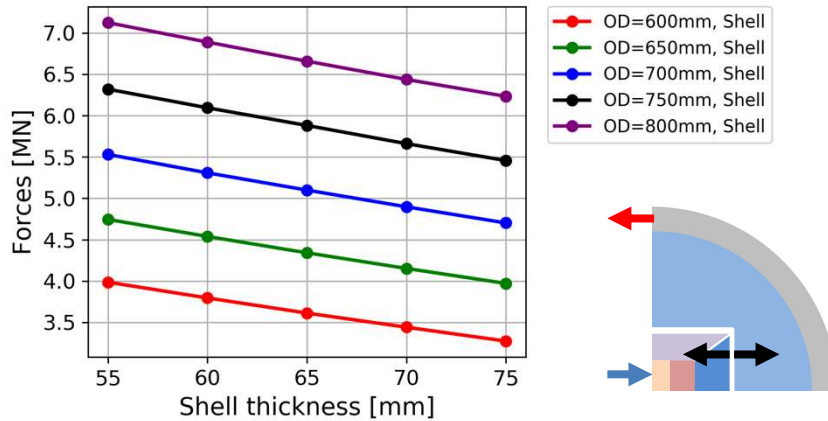
- **Currently considered for calculation**
  - Coil pack width 320 mm
  - Shell OD 750 mm
  - Shell TH 75 mm
- **Current configuration allows pre-tensioning the shell to ~12 MN/m force per magnet quadrant**
  - Assuming that yoke remains open
  - Assuming max. 45 MPa bladder pressure
  - Maximum forces defined but the shell OD
- **Factors needed to be consider**
  - Hybrid HTS/LTS coils (coil pack size)
  - Cryostat (Shell OD)
  - Closed yoke gap (structure rigidity)





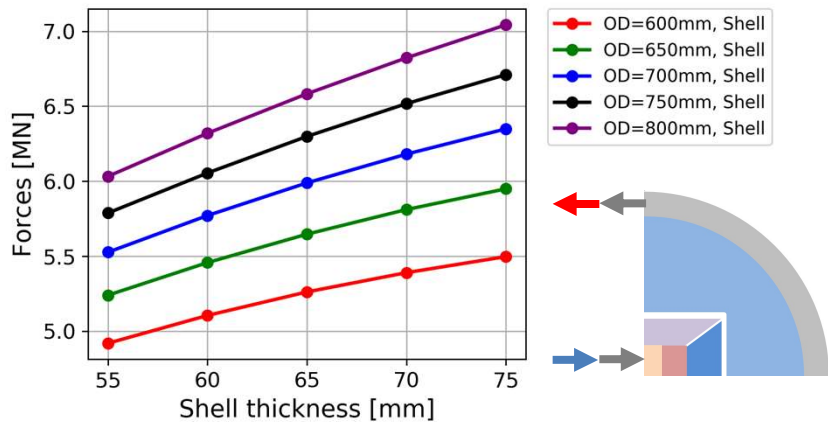
# Maximum reaction forces in the shell

Maximum force at room temperature

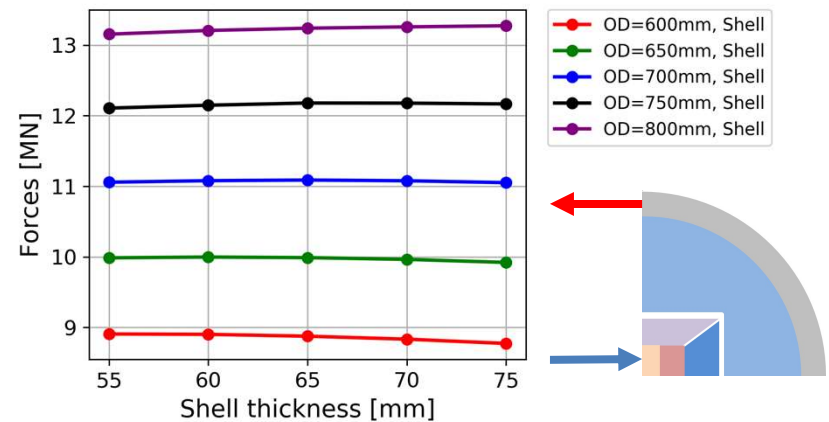


- Maximum force defined by OD
- Thicker shell
  - Less space for bladder
  - Thinner shell can be inserted with fixed bladder pressure
  - Higher force gained during cool-down

Maximum force due to cool-down



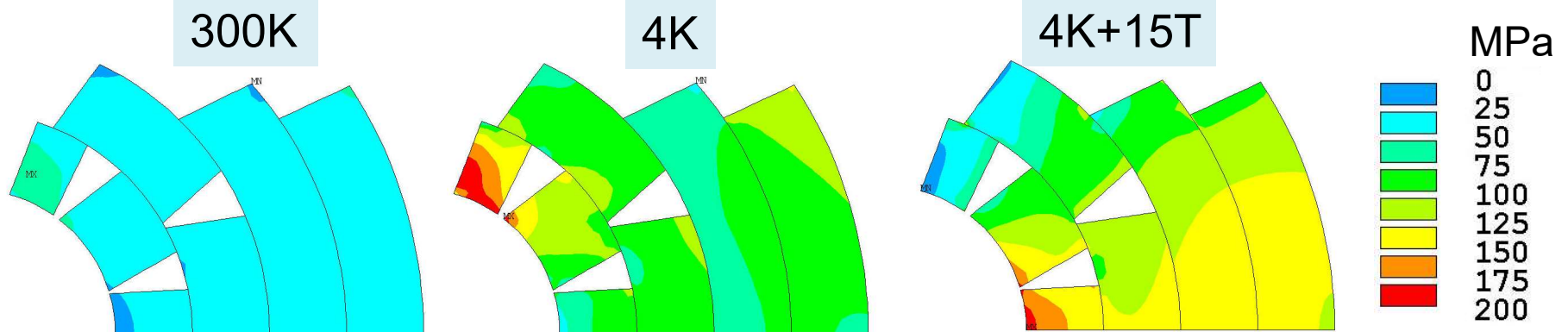
Maximum reaction force in the shell





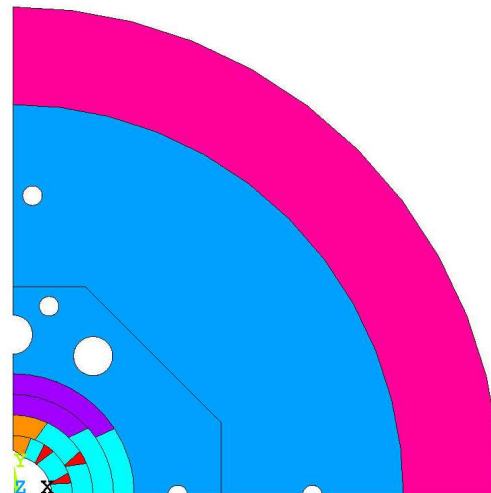


# Utility Structure 2D analysis

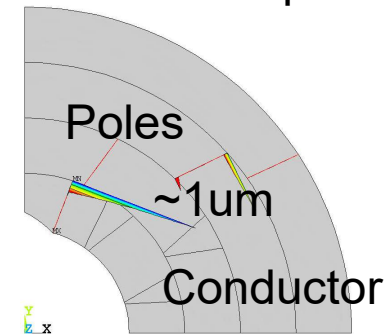


- **15 T Pre-load**
  - Bladder operation < 80 MPa
  - Room temperature < 60 MPa
  - Cool-down < 200 MPa
  - Magnetic forces < 200 MPa
- **16 T Pre-load**
  - Bladder operation ~ 100 MPa
  - Room temperature ~ 80 MPa
  - Cool-down ~ 220 MPa
  - Magnetic forces ~ 220 MPa

Updated cross-section



Pole-turn separation

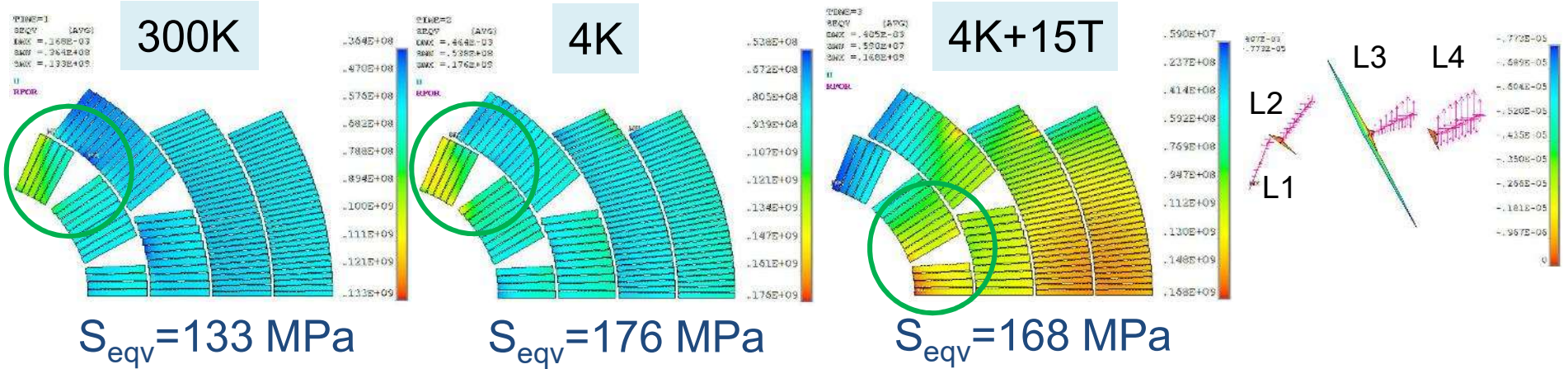


	4 K	300 K
$E_r$	55 GPa	44 GPa
$E_\theta$	44 GPa	44 GPa
$\alpha_r$	2.6 x10 <sup>3</sup> 1/K	
$\alpha_\theta$	3.3 x10 <sup>3</sup> 1/K	

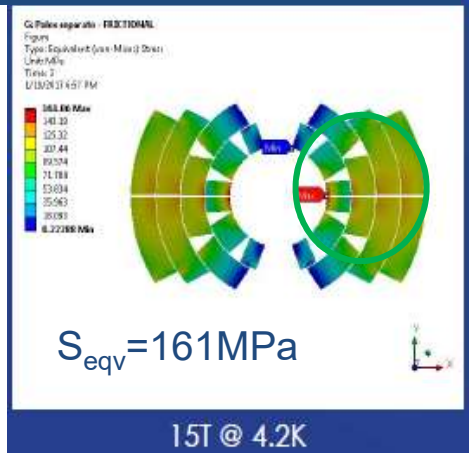
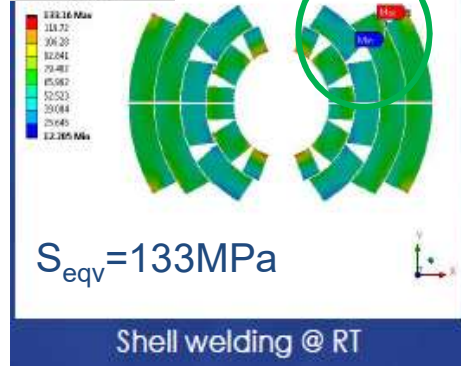
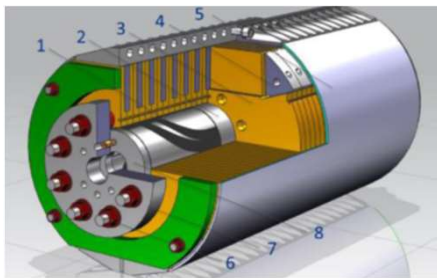


# 15 T dipole demonstrator - 2D FEA

## FNAL analysis

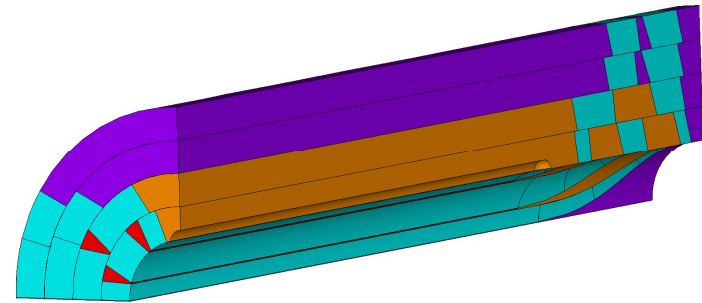
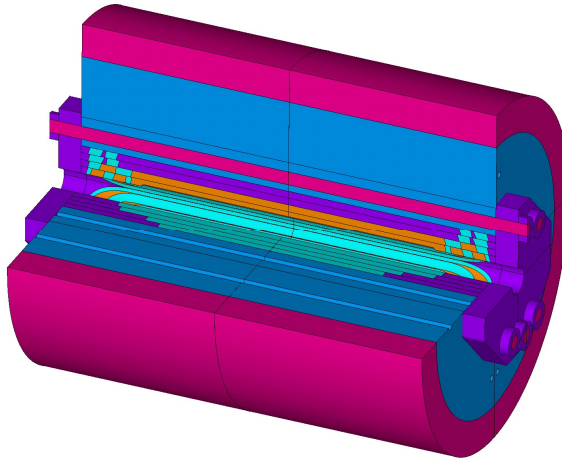


## FEAC/U. Patras 2D analysis

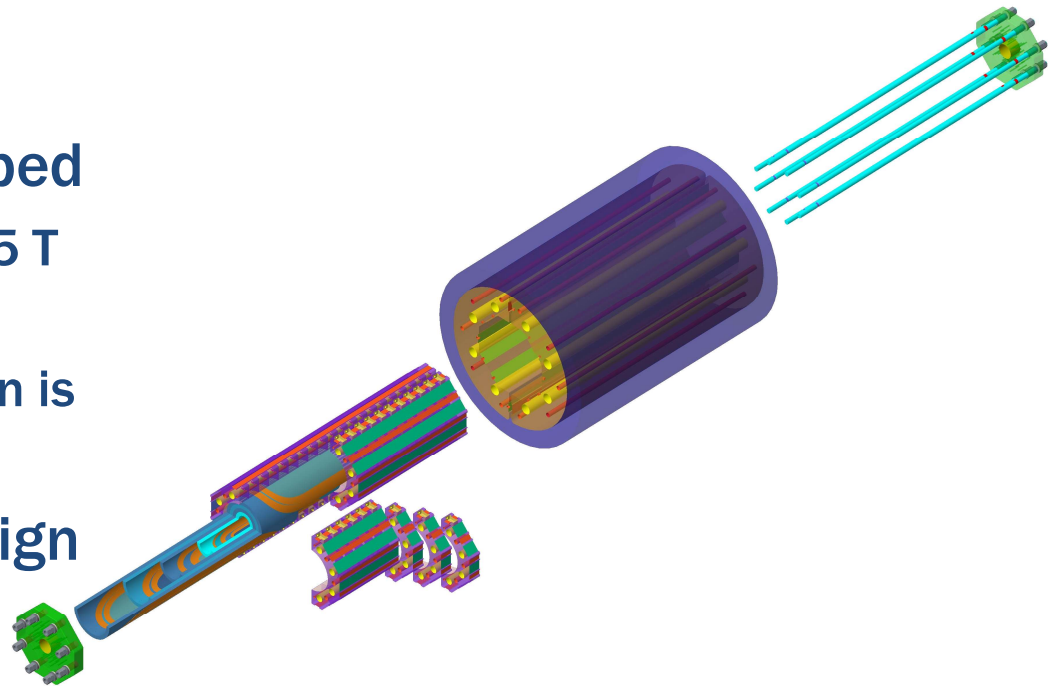




## Development of the 3D model



- **3D ANSYS model developed**
  - Initial simulations using 15 T Cos-Theta coils
  - Validation and optimization is ongoing
- **Work on engineering design was initiated**

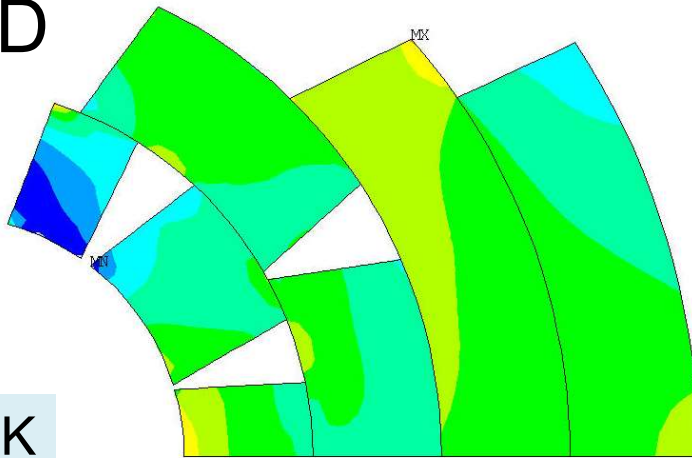




# Utility Structure

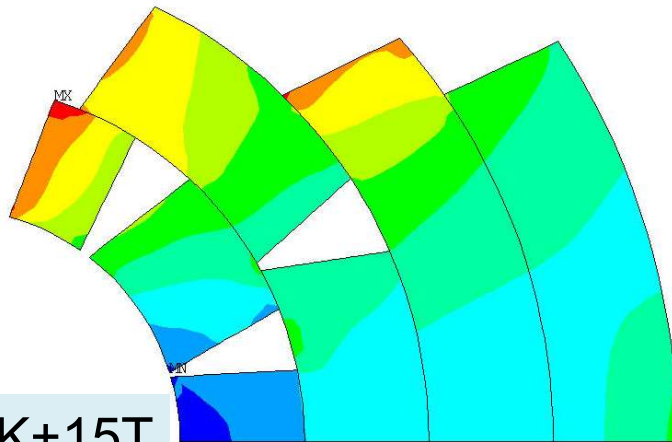
## Azimuthal coil stress (2D vs 3D)

2D

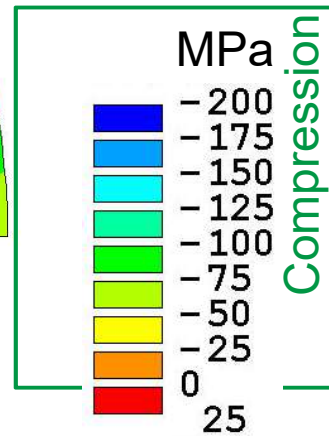
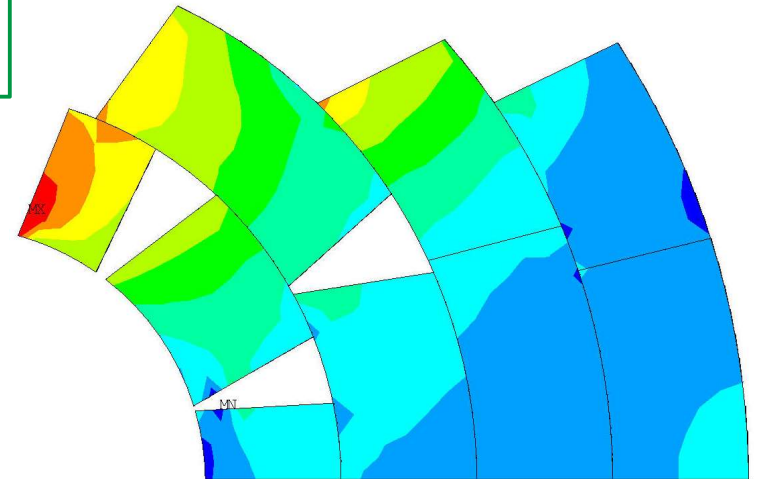
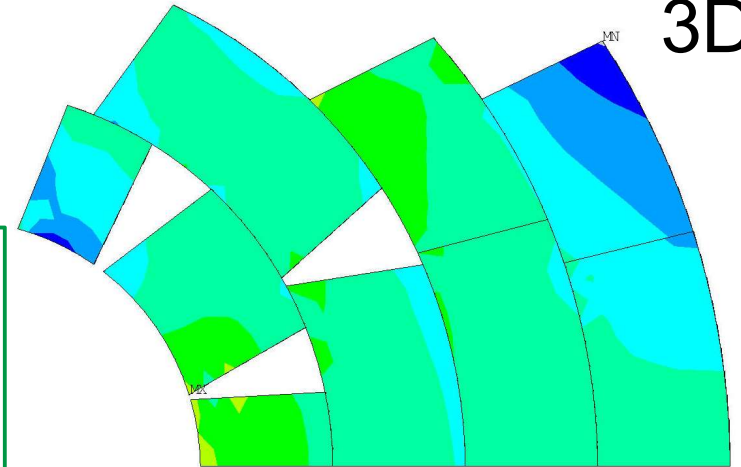


4K

4K+15T



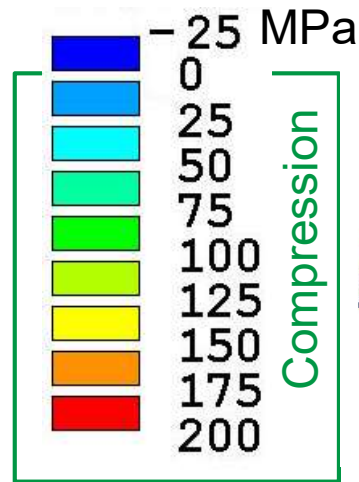
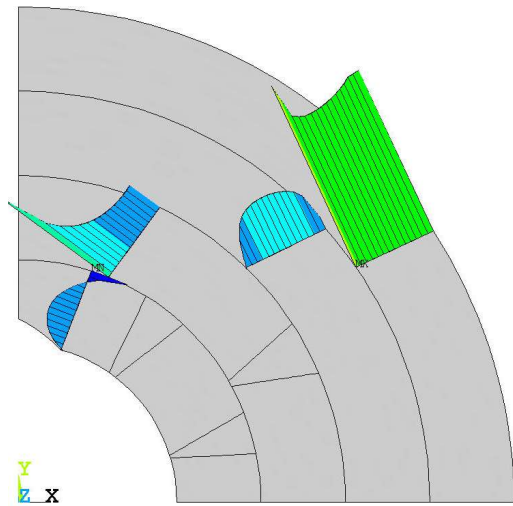
3D



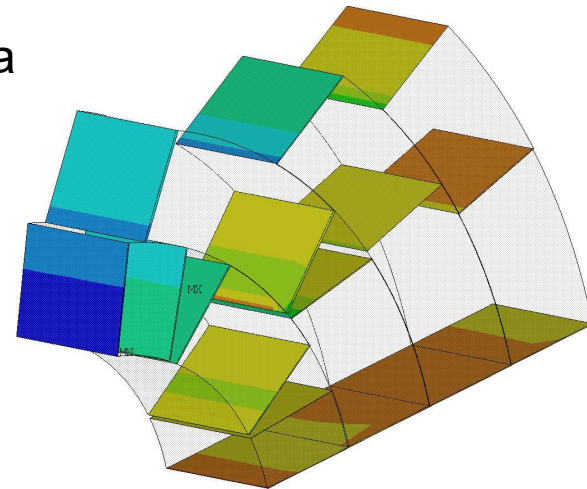


# Coil to pole/spacer contact

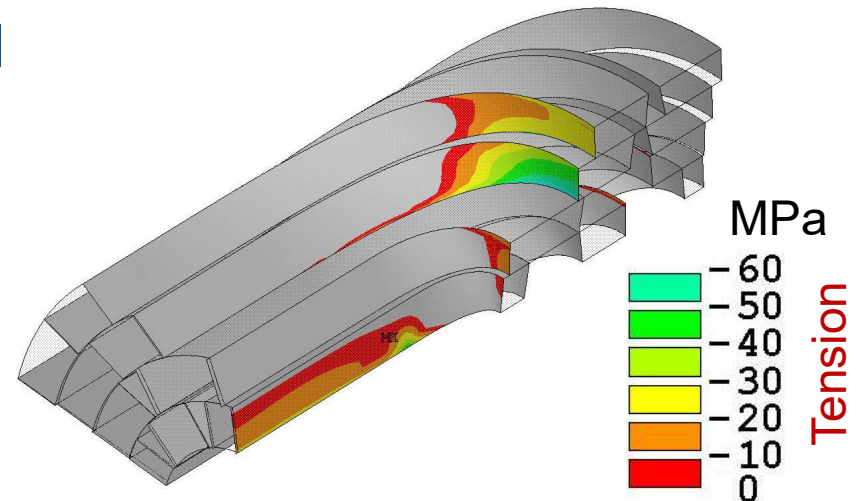
2D



3D

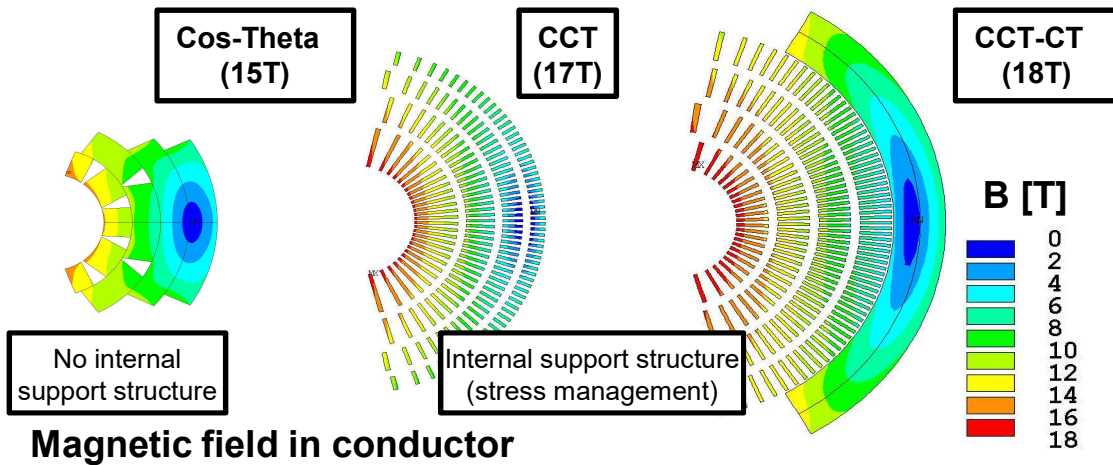
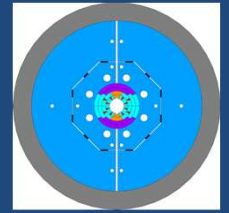


- Optimization of the pre-load level
  - Orthotropic coil properties
- Axial pre-load and tension in the coil-ends under investigation
  - Copper wedge to spacer interface
  - Layer 3-4 to pole contact

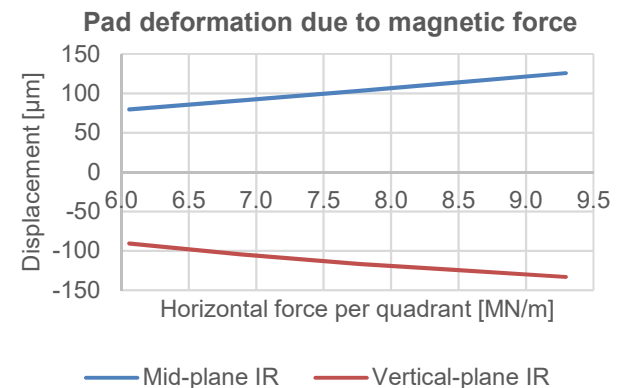
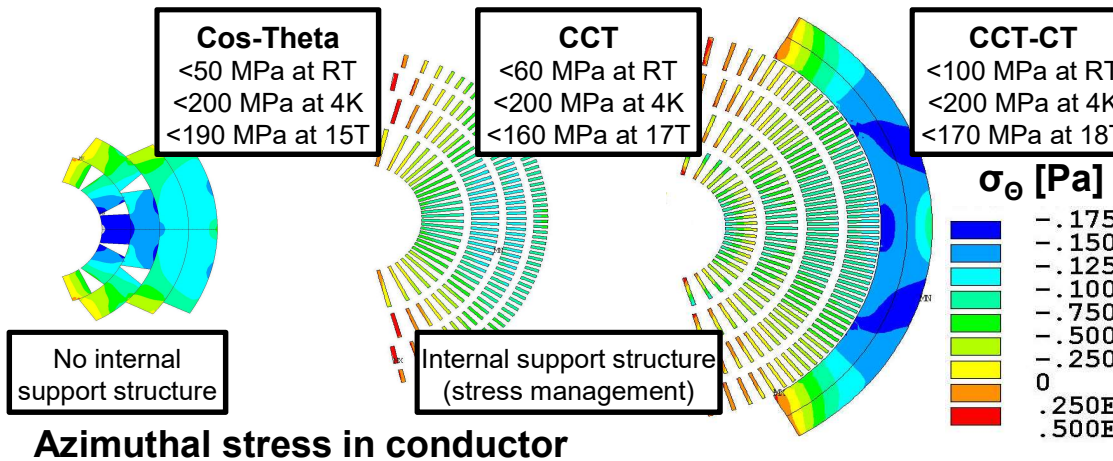




# Utility Structure With different coil types



- Same structure was applied
- Shell OD 750mm
- Shell TH 75mm
- Same iron yoke
- Only pads were changed
- Pre-load adjusted only by load-key shims
- Possibility to close the yoke gap at cold (rigidity)

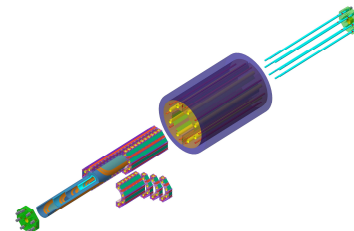
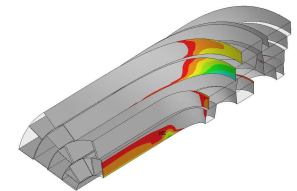
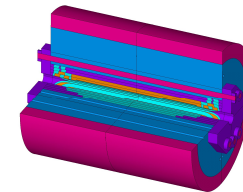
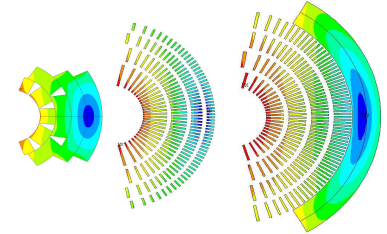
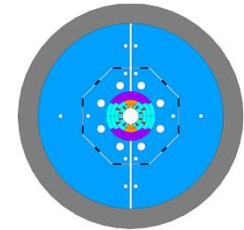


- Closed collars ?



# Conclusions and Future Work

- **Concept of the utility structure has been developed and analyzed**
  - Reusable yoke-shell assembly with a coil-dedicated pads
  - Tentative dimensions established but further optimization is needed
  - 15 T Cos-theta coil and CCT coil analyzed with utility structure pre-load
  - Work on 16T coil pre-load analysis initiated (CT/SM, CCT/CT, LTS/HTS hybrids)
  - Structure rigidity with closed yokes needs further investigation (closed collars)
- **3D model of the utility structure developed**
  - 15 T Cos-theta coil model implemented
  - Validation and optimization is ongoing
  - Axial pre-load system implemented
- **Engineering design work initiated**
  - CAD model of the full structure
  - Assembly procedure
  - Short mechanical mock-up





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**Thank You!**