



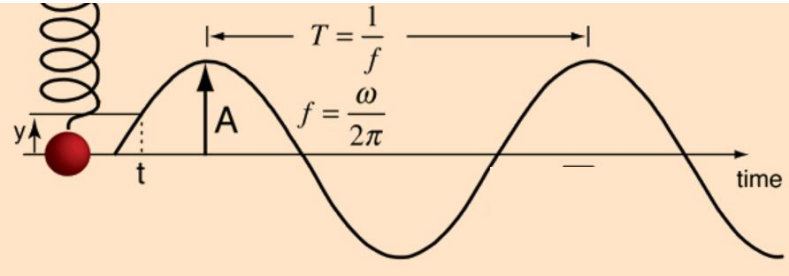
# Vibrations in (superconducting) coils

Stoyan Stoynev /FNAL/

MDP meeting

6 January 2021

# Preface 1: Periodic motion



$$y = A \sin \omega t$$

$$v = \omega A \cos \omega t$$

(vibration velocity,  
time derivative of  $y$ )

$$a = -\omega^2 A \sin \omega t = -\omega^2 y$$

(vibration acceleration,  
double time derivative of  $y$ )

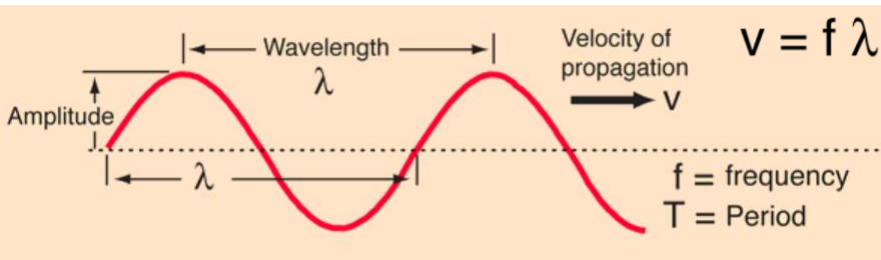
Relation between quantities at extremes:

$$x_{\max} = A \quad (x \equiv y)$$

$$v_{\max} = A\omega$$

$$a_{\max} = A\omega^2$$

# Preface 1: Wave power relations



Power transmitted by string wave

$$= \frac{1}{2} \mu \omega^2 A^2 v$$

Labels in the equation:

- $\mu$ : Mass per unit length of the string
- $\omega$ : Angular frequency of the wave
- $A$ : Wave amplitude
- $v$ : Wave propagation velocity

$$A = \sqrt{\frac{2P}{v\mu}}$$

$$v \sim 4 \times 10^3 \text{ m/s}$$

$$\mu \sim 0.5 \text{ kg/m}$$

( $P/\mu$  remains the same for a single “string” or a bunch of “strings” if all of them are “powered”)

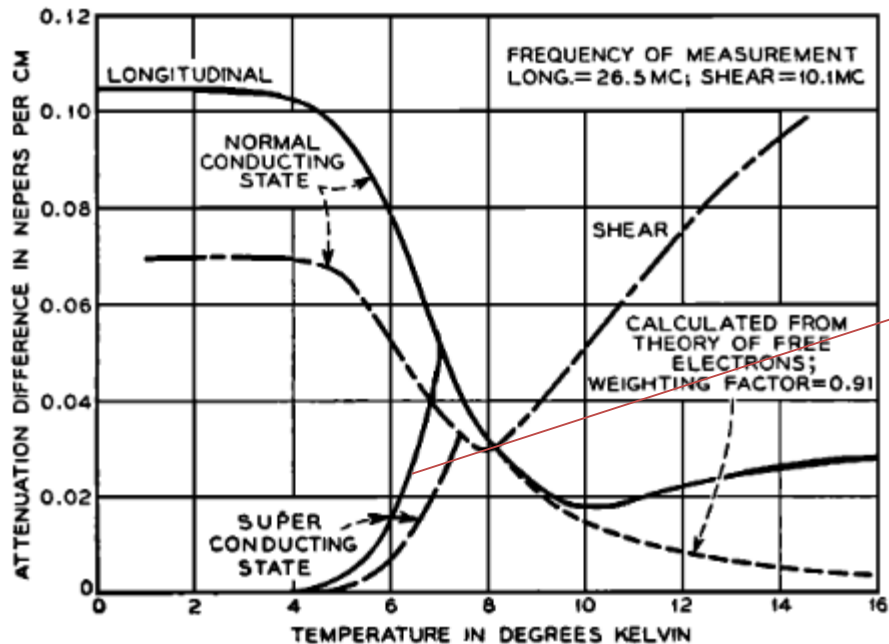
- Let’s assume the cable of a coil can serve as an acoustic transmission line (at least initially), that is it does not “radiate” energy
- Let’s see what amplitudes we can achieve along this “string” with some set of parameters

P (W)	w (1/s)	A (m)	Aw (m/s)	$\lambda$ (m)
10	$10^5$	$10^{-6}$	0.1	0.25
100	$10^5$	$3.1 \times 10^{-6}$	sqrt(0.1)	0.25
1000	$10^5$	$10^{-5}$	1	0.25
10	$5 \times 10^5$	$2 \times 10^{-7}$	0.1	$5 \times 10^{-2}$
100	$5 \times 10^5$	$6.2 \times 10^{-7}$	sqrt(0.1)	$5 \times 10^{-2}$
1000	$5 \times 10^5$	$2 \times 10^{-6}$	1	$5 \times 10^{-2}$
1000	$5 \times 10^6$	$2 \times 10^{-7}$	1	$5 \times 10^{-3}$

# Preface 1: More realistic conditions

- Coils are not strings, there is more complicated energy transfer through the bulk and interfaces
  - This doesn't invalidate all string projections but there are apparent limitations
- Attenuation – it is complicated

Falls ~ linearly with frequency, falls with temperature except for very low temperatures, superconductors become “transparent” at low temperatures, magnetic field decreases attenuation, multi-material entities (like Nb<sub>3</sub>Sn+Cu) have features,...

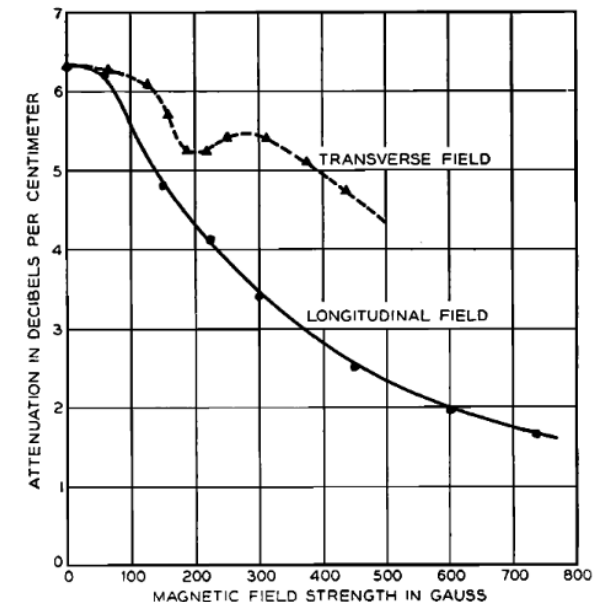


$$\frac{\alpha_{sc}}{\alpha} \sim \frac{u}{e^v T_c/T + 1}$$

(away from  $T_c$ ),  $u, v$  are  $\sim 1-10$

$\alpha$  is attenuation coefficient

$$A = A_0 \exp(-\alpha x).$$



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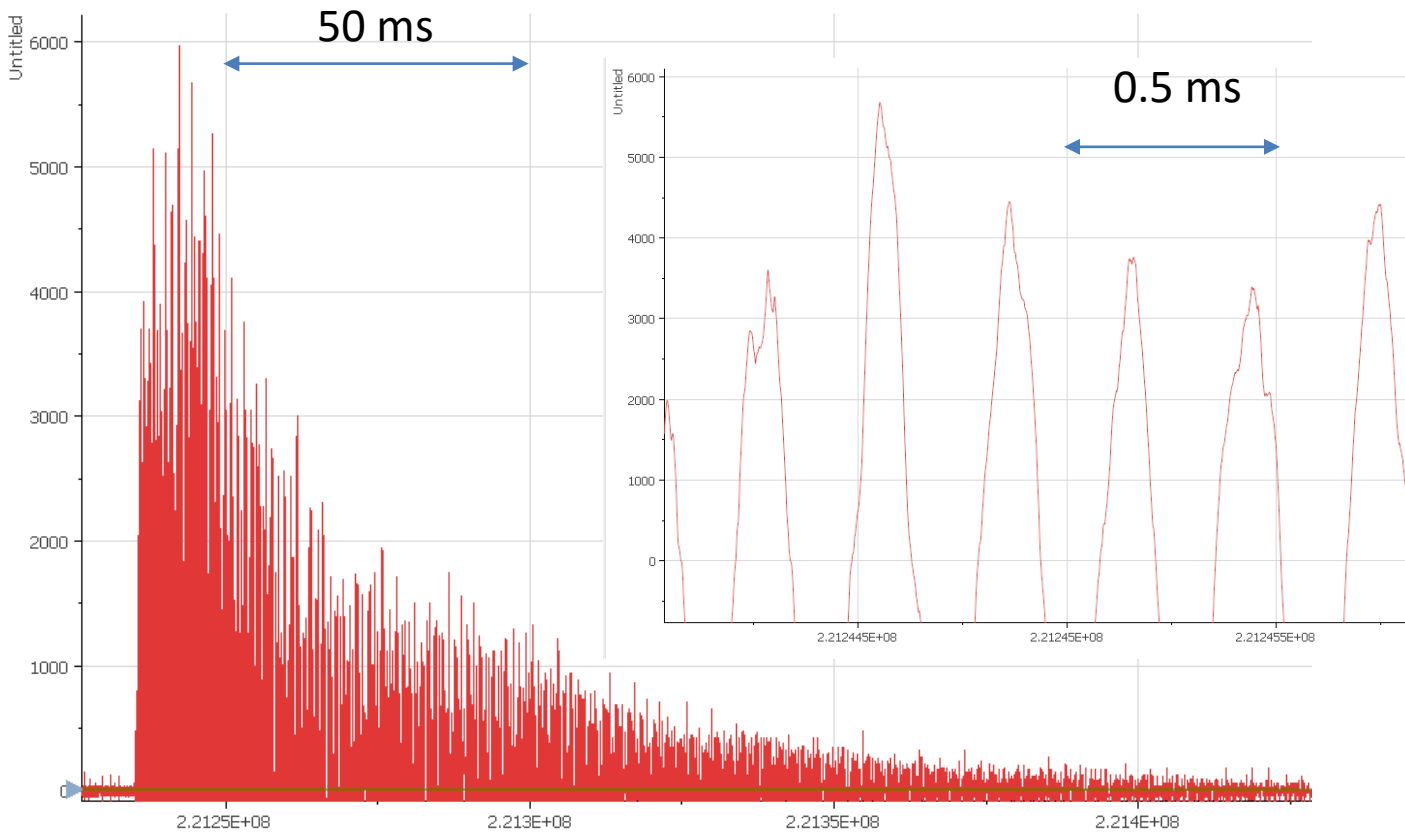
SEPTEMBER, 1956

Ultrasonic Attenuation at Low Temperatures for Metals in the Normal and Superconducting States

W. P. MASON AND H. E. BÖMMEL  
Bell Telephone Laboratories, Incorporated, Murray Hill, New Jersey



# Preface 1: Acoustic attenuation in SC magnets



In first approximation we can accept an ultrasonic wave will not get attenuated too fast in the coil (it takes at least tens of periods to drop significantly).

In “15 T” magnet tests we see the amplitude drops by  $\sim 1/3$  in  $\sim 30$  ms. The periodic pattern is associated to the length of the magnet (1 m  $\rightarrow \sim 0.3$  ms period). So it drops by  $1/3$  for 100 m ( $\sim 4$  kHz) which is  $\sim 0.1$  dB/m.

At room temperature I find this formula:  
 $\alpha \approx C_d \times f$ ,  $C_d \sim 30$  dB/m/MHz (steel/Cu; Al is  $\sim 10$ )  
and thus  $\alpha \approx 30 \times 4 \times 10^{-3} = 0.12$  dB/m.  
This is the same order of magnitude as above (though different temperatures).



Review

**A Comprehensive Report on Ultrasonic Attenuation of Engineering Materials, Including Metals, Ceramics, Polymers, Fiber-Reinforced Composites, Wood, and Rocks**

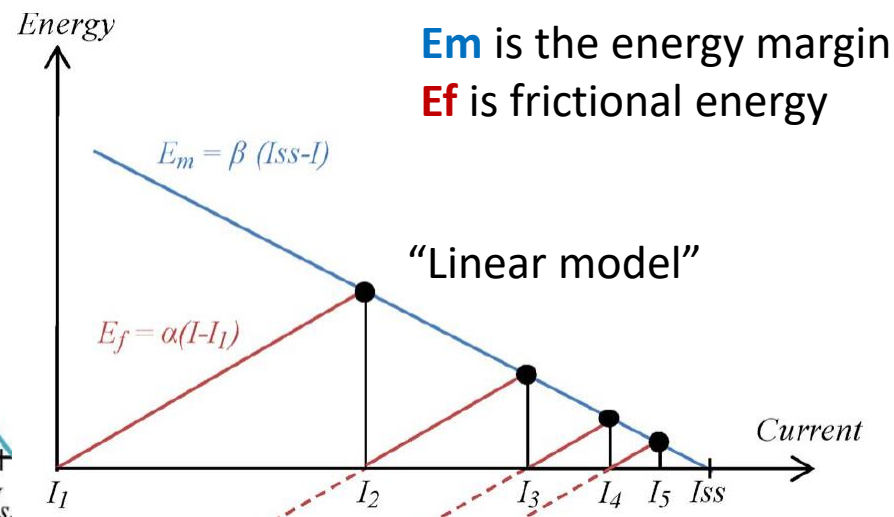
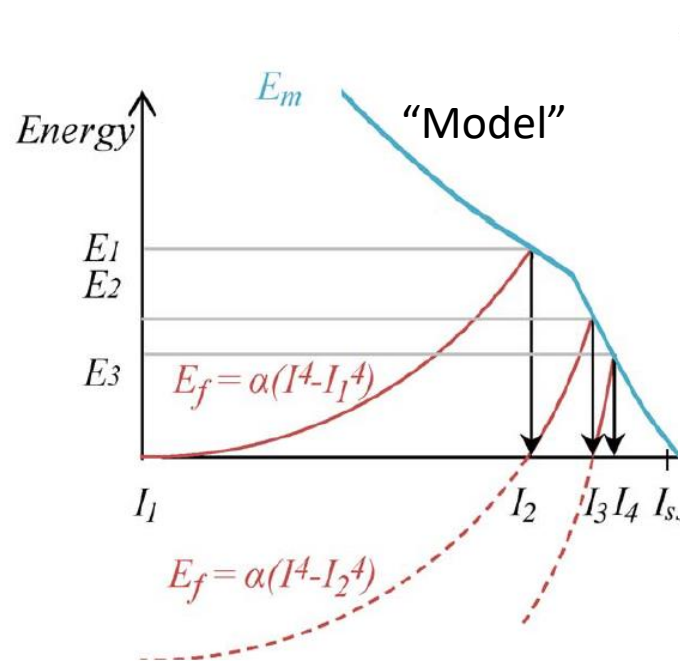
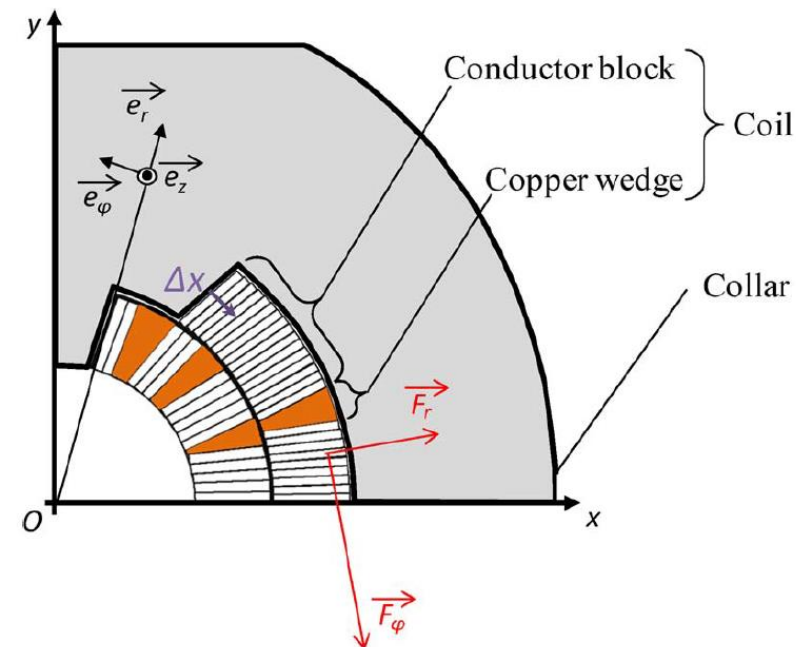
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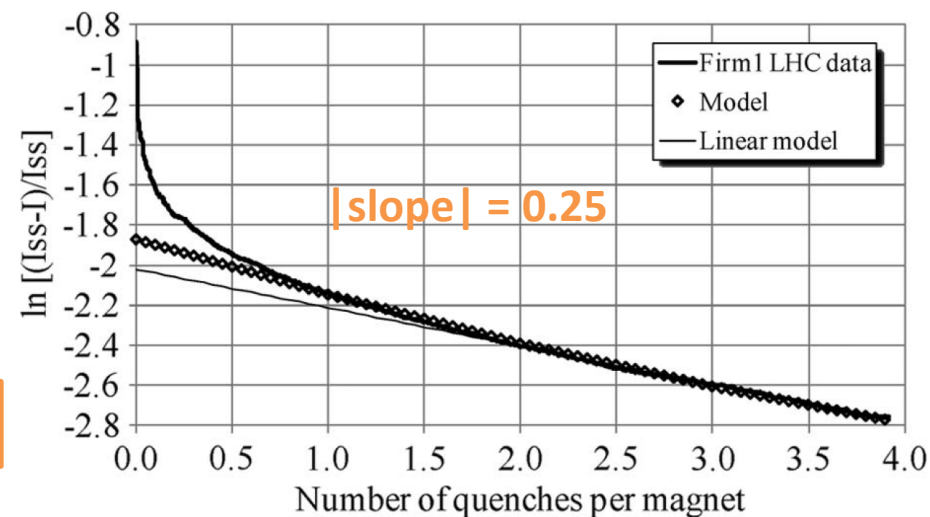
# Preface 2: Slip-Stick in (LHC) magnets



$$I_{ss} - I_{n+1} = (I_{ss} - I_1) \left(1 + \frac{\beta}{\alpha}\right)^{-n}$$

$$\ln\left(1 - \frac{I_{n+1}}{I_{ss}}\right) = \ln\left(1 - \frac{I_1}{I_{ss}}\right) - n \ln\left(1 + \frac{\beta}{\alpha}\right)$$

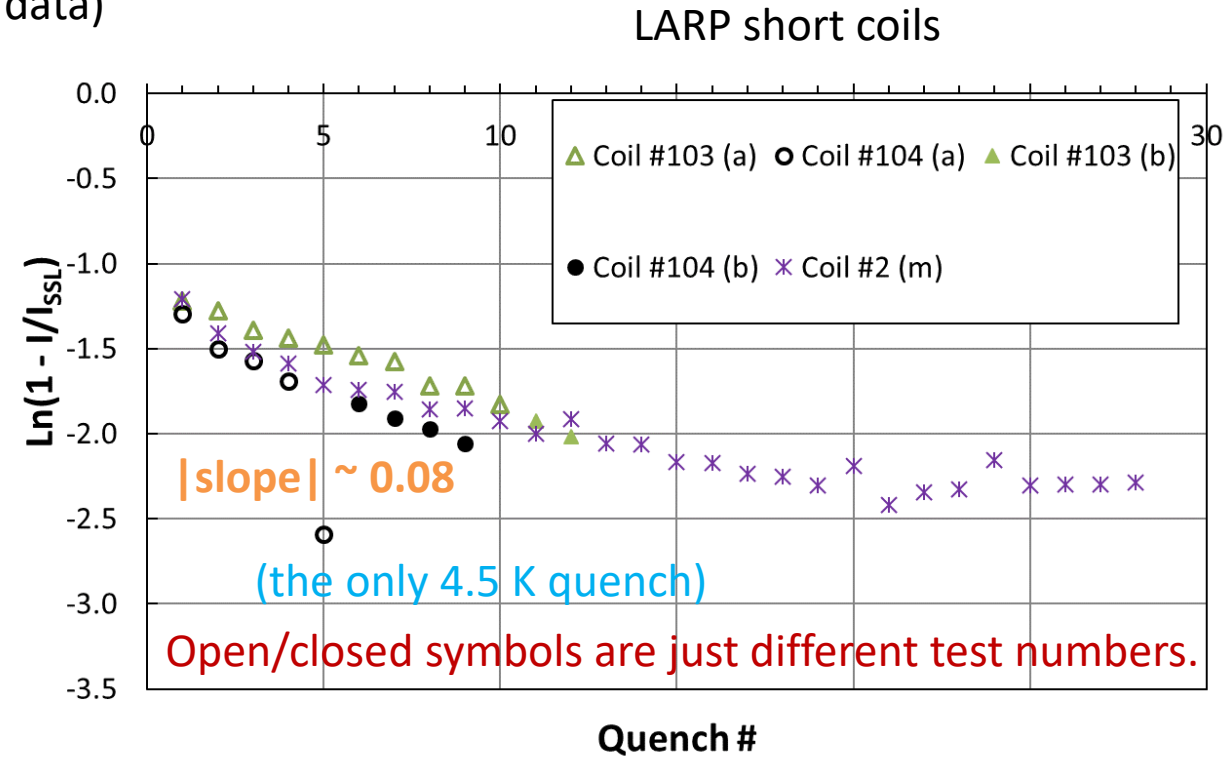
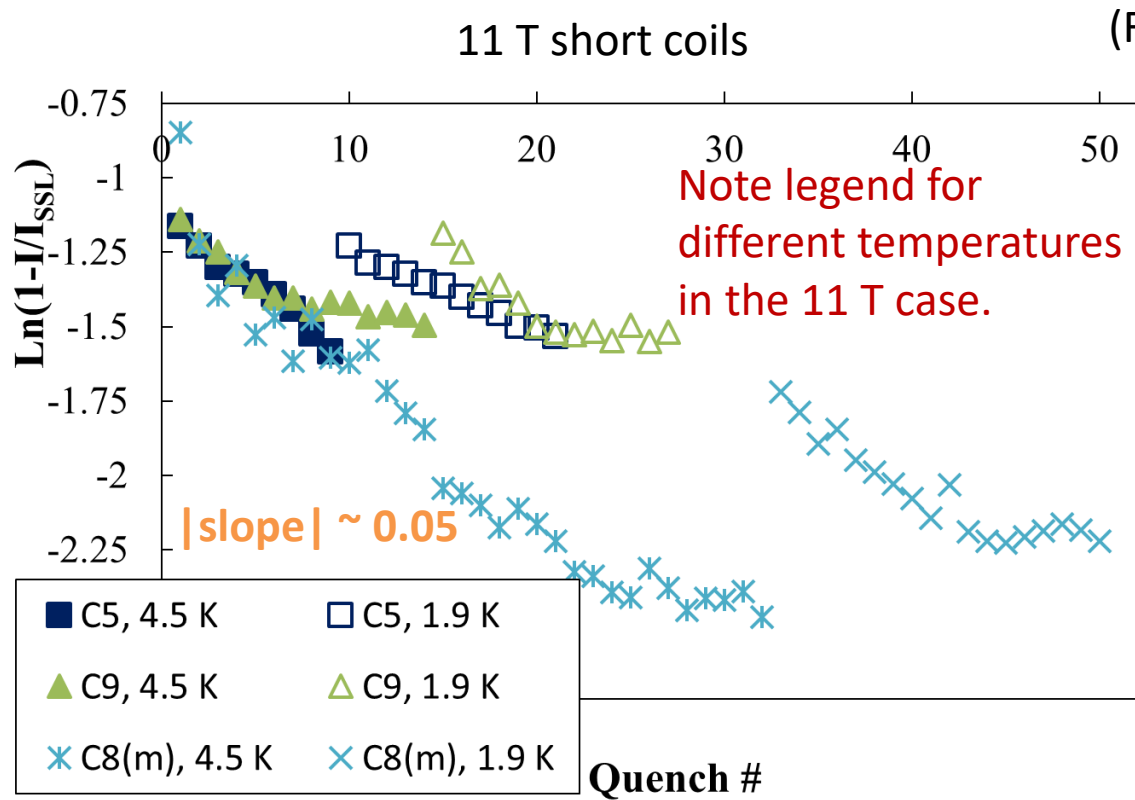
**|slope|**



Slip-Stick Mechanism in Training the Superconducting Magnets in the Large Hadron Collider

Pier Paolo Granieri, Clément Lorin, and Ezio Todesco

# Preface 2: Analogous 11 T and LARP s-quad curves



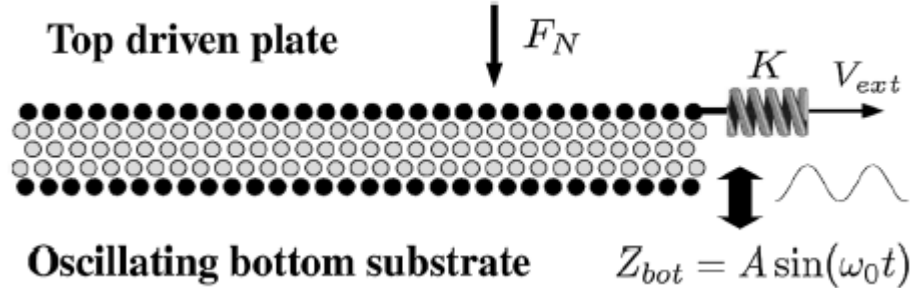
The flattening of curves mean coils reaching a training limit (at certain conditions)

# Vibration, friction, magnetic field



# Non-rigid interface

Rigid surfaces confining non-rigid material  
(you can think metal and insulation material)



PRL 103, 085502 (2009)

PHYSICAL REVIEW LETTERS

## Suppression of Friction by Mechanical Vibrations

Rosario Capozza,<sup>1</sup> Andrea Vanossi,<sup>2,1</sup> Alessandro Vezzani,<sup>1,3</sup> and Stefano Zapperi<sup>1,4</sup>

- In first approximation there are three forces acting in vertical direction:
  - ✓ Applied normal force,  $F_N$
  - ✓ Inertial force (from vibrations),  $F_{in}$
  - ✓ Damping force (energy dissipation),  $F_{damp}$

Let's  $M = M_p + M_{top}$  be the total mass of the non-rigid and the top rigid material.

$F_{in} = M\ddot{Z}_{bot}$  and then the **maximal force** (of oscillation) is  **$MA\omega_0^2$** . The damping force is proportional to velocity (of oscillation)  $F_{damp} = M_p \eta \dot{Z}_{bot}$  ( $\eta$  is damping coefficient) and then its **maximum** is  **$M_p \eta A \omega_0$** . At frequency  $\omega_1$  the inertial force overcomes the other forces and friction (proportional to the normal down-force) vanishes.

$$MA\omega_1^2 = F_N + M_p \eta A \omega_1. \quad \text{Using dimensionless variables} \quad \tilde{f} \equiv \frac{F_N}{MA\eta^2} \quad \tilde{m} \equiv \frac{M_p}{M} \quad \tilde{\omega} \equiv \frac{\omega}{\eta}.$$

one gets 
$$\tilde{\omega}_1 = \frac{1}{2}(\tilde{m} + \sqrt{\tilde{m}^2 + 4\tilde{f}}).$$

**Friction above oscillation frequency  $\omega_1$  is suppressed.**

# Non-rigid interface (2)

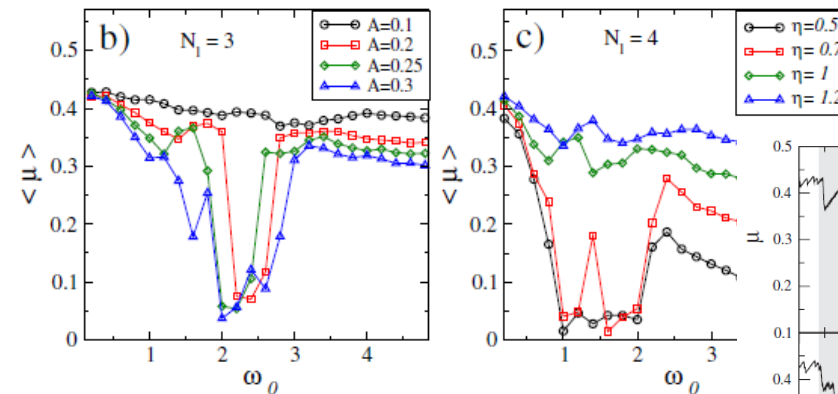
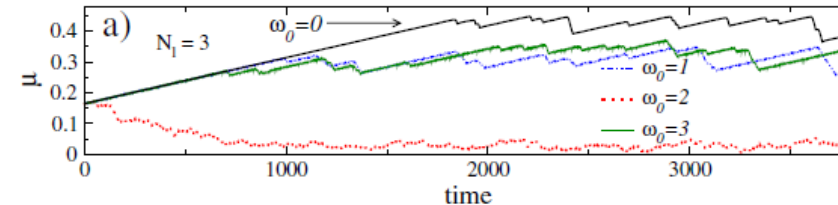
PRL **103**, 085502 (2009)

PHYSICAL REVIEW LETTERS

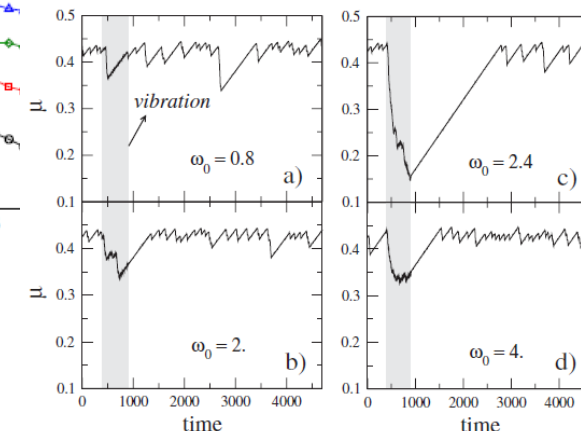
## Suppression of Friction by Mechanical Vibrations

Rosario Capozza,<sup>1</sup> Andrea Vanossi,<sup>2,1</sup> Alessandro Vezzani,<sup>1,3</sup> and Stefano Zapperi<sup>1,4</sup>

Friction coefficient vs time and  
time averaged coefficient vs  $w_0$  in numerical simulations



Effect of a short time  
vibration pulse on the  
stick-slip behavior



Numerical simulations confirm the findings as well as  
later experimental reports (other authors).

- During oscillations particles are subject to forces
- There is a characteristic time  $\Delta t$  for their action expressed in the impulse formula  $F\Delta t = m\Delta v$
- In our case:  

$$\Delta t \simeq \dot{Z}_{\text{bot}} M / F_N \simeq A \omega_0 M / F_N$$
- If the period of oscillation is smaller than this characteristic time ( $2\pi/\omega_0 < \Delta t$ ) there is no effective transfer of oscillation energy between particles.
- Explicitly:  

$$\tilde{\omega}_2 = \sqrt{2\pi\tilde{f}}$$
- Above  $w_2$  friction is no longer effectively suppressed
- Thus, friction is suppressed if  $w_1 < w_0 < w_2$

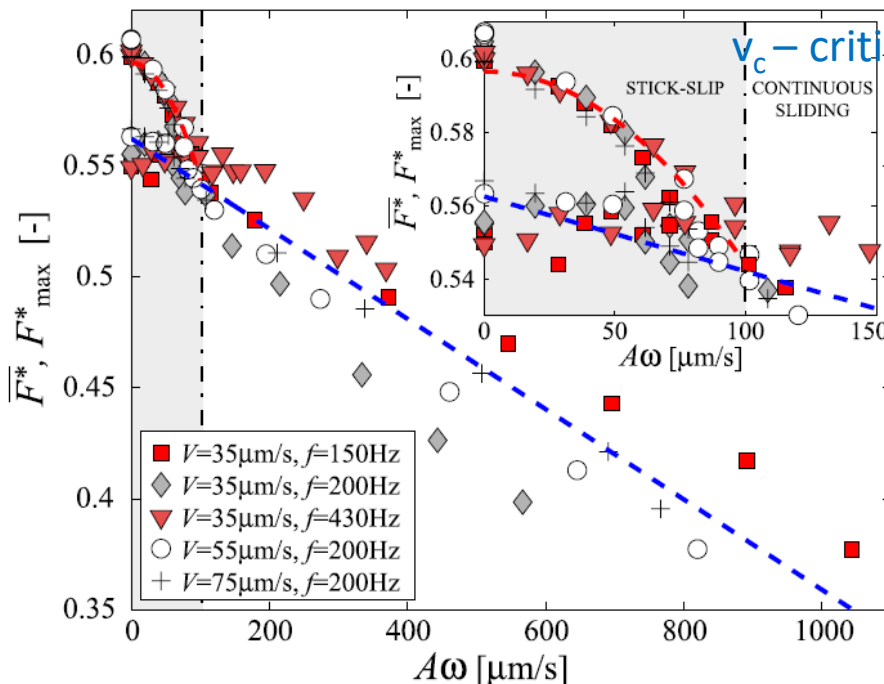
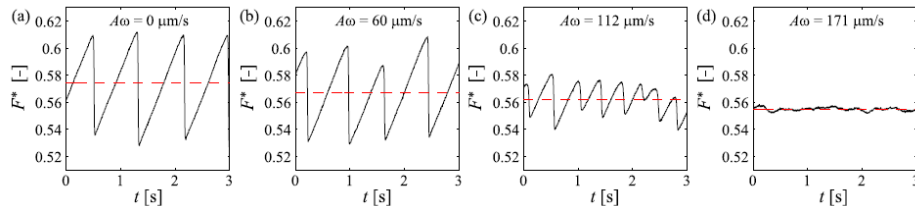
# Granular surfaces

[www.nature.com/scientificreports](http://www.nature.com/scientificreports)

## Granular friction: Triggering large events with small vibrations

Henri Lastakowski, Jean-Christophe Géminard & Valérie Vidal

Normalized tangential force (friction coefficient)  
vs time at different vibration velocities and vs vibration velocity



$v_c$  – critical velocity

Beyond  $v_c$  slip-stick transitions to continuous sliding.  
The velocity dependence is direct  
(no hidden dependences to other parameters).

$F^*$  is the driving force  
normalized to pressing  
(normal) force or effectively  
– the friction coefficient.

Experiments confirm those findings.

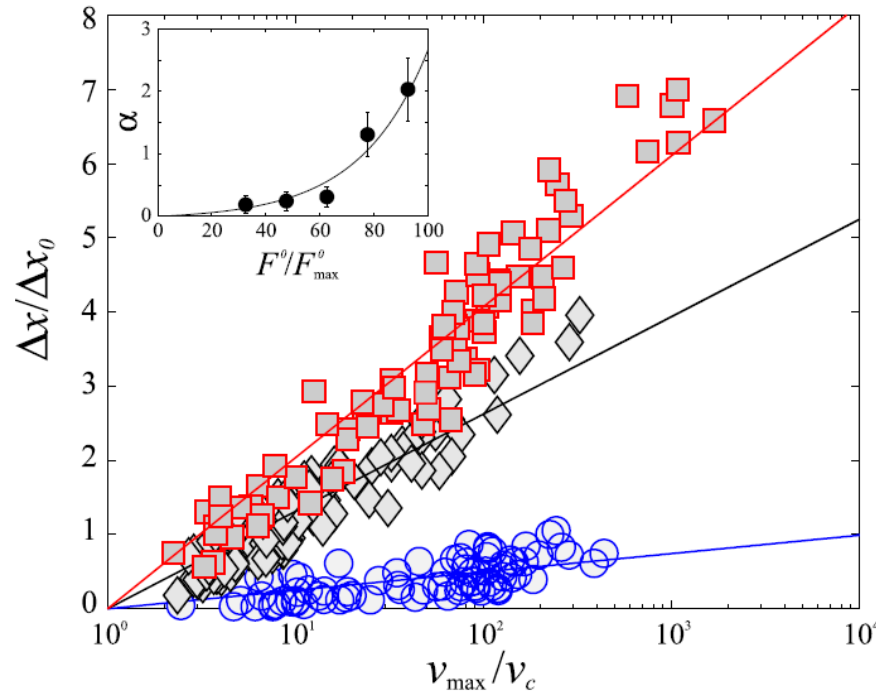
# Granular surfaces (2)

[www.nature.com/scientificreports](http://www.nature.com/scientificreports)

## Granular friction: Triggering large events with small vibrations

Henri Lastakowski, Jean-Christophe Géminard & Valérie Vidal

Displacement after single mechanical disturbance (from static case)  
Normalization is to the case with no disturbance ( $x_0$ )  
and critical velocity  $v_c$



$\alpha$  here is the slope  
of  $\Delta x/\Delta x_0$

$F^0/F^0_{\max}$  on the plot is  
(0.85,1) - red  
(0.7,0.85) - black  
(0.45,0.55) - blue

$F^0_{\max}$  is when slip occurs  
without vibrations

As before, the vibrational velocity ( $A\omega$ ) is the parameter  
of interest. Higher  $A\omega$  higher the released energy ( $\Delta x$ )  
from a slip.

If a barrier with size  $\xi_c$  needs to be overcome  
to “unstuck” movement:

$$(\rho d^3 g) \xi \sim \rho d^3 (A\omega)_c^2$$

Potential energy                      Kinetic (inertial) energy

$$\xi_c \sim (A\omega)_c^2 / g$$

$g$  is characteristic acceleration of the normal force

Indeed, critical velocity was found to correlate well with surface granularity independently on vibration type  
(authors point out that high pressure may require high  $A$  explicitly).

# Summary 1

- Studies show that vibrations in any direction with respect to friction force have similar characteristics although quantitatively there are some (small) differences
- In sliding over non-rigid material one can expect a window of vibration frequencies ( $w_1, w_2$ ) where friction is suppressed

$$w_1 < w < w_2$$

- Granular “imperfections” induce a characteristic critical (friction) velocity related to their size

$$\xi_c \sim (A\omega)_c^2 / g$$

- Other authors point out that rearrangements in the frictional media (which affects friction) can be induced when the mechanical wavelength is of the order of the rearrangement scale



# Context: coils

- Coils can be subjected up to 200 MPa due to Lorentz force and pre-stress
  - Let's take 100 MPa normal to the cable wide side (2 cm cable width, for instance)
  - Let's use 5 cm as a characteristic test scale over the cable length, then  $F_N = 100 \text{ kN}$
  - The coil mass (all turns) over that area is  $M \sim 5 \text{ kg}$

Just for convenience  
(can work with pressure alone)

Damping force:  $M_p \eta A w_0$   
(non-rigid interface)

$Aw$  is  $< 1$  (in SI) for us,  $M_p$  (cable insulation for the above  $M$ ) is a very small number, the damping constant could not be large, or we wouldn't hear anything during magnet quenching;

**at the end, this term is negligible with respect to large forces in the magnet**

$w_1$  and  $w_2$ :

(non-rigid interface)

$$w_1 = \sqrt{F_N / (MA)} \quad \text{and} \quad w_2 = \sqrt{2\pi} w_1$$

If we set  $A = 3 \text{ microns}$ ,  $w_1 \sim 10^5 \text{ Hz}$  (same as  $f_1 \sim 17 \text{ kHz}$ ), which we can achieve with  $\sim 100 \text{ W}$  of power according to an earlier table.

$$w_2 \sim 2.5 \times 10^5 \text{ Hz}$$

...and higher  
 $Aw$  higher  
the released  
energy at slip

$g, \xi_c$ :

(rigid granular barriers)

$$g = F_N / M \sim 20 \text{ km/s}^2 \quad \xi_c \sim (A\omega)_c / g$$

Within reasonable constraints we can achieve at most  $Aw = 1 \text{ m/s}$ .

Thus, we can "resolve" barriers at scale  $\xi_c < M / F_N < 50 \text{ microns}$

$\lambda$ :

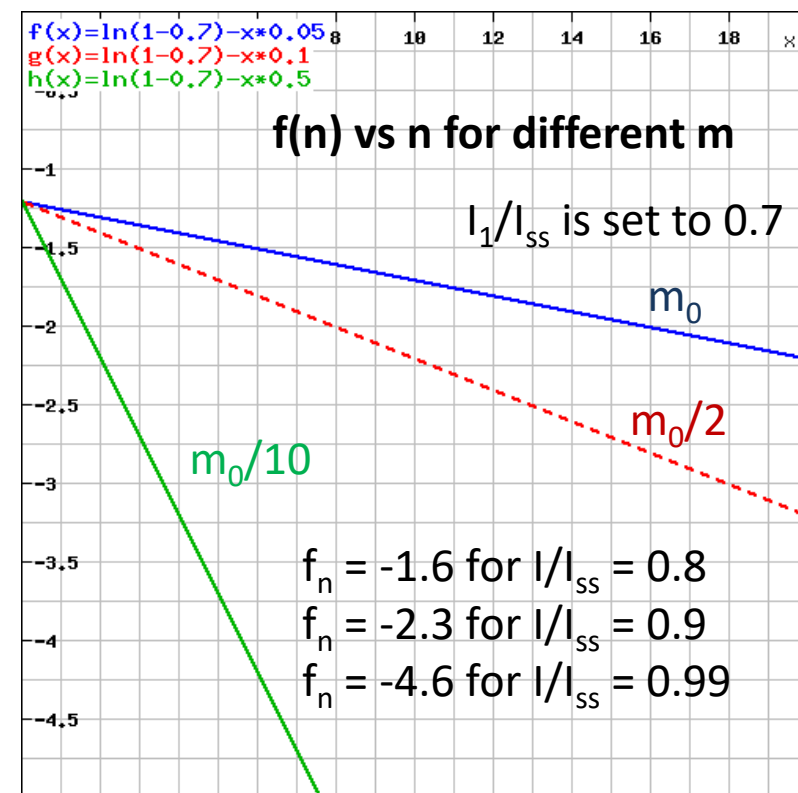
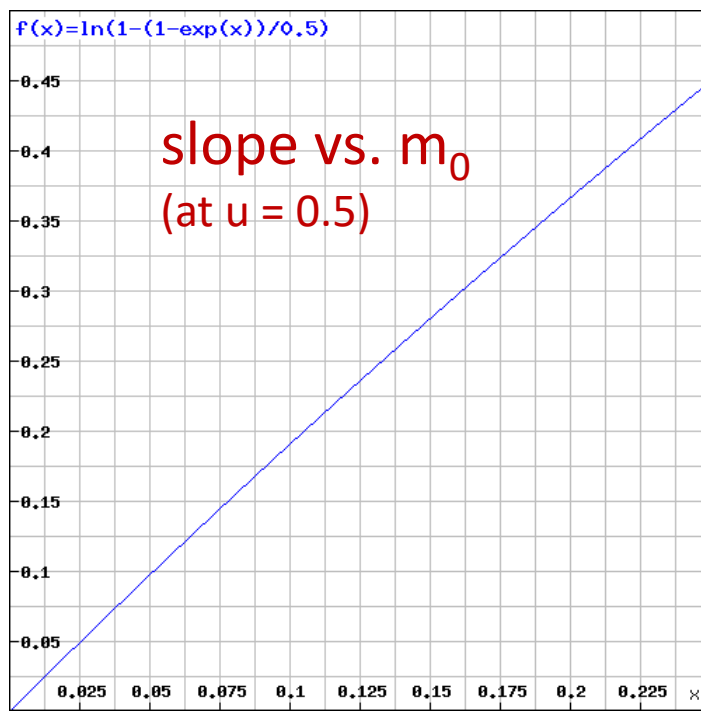
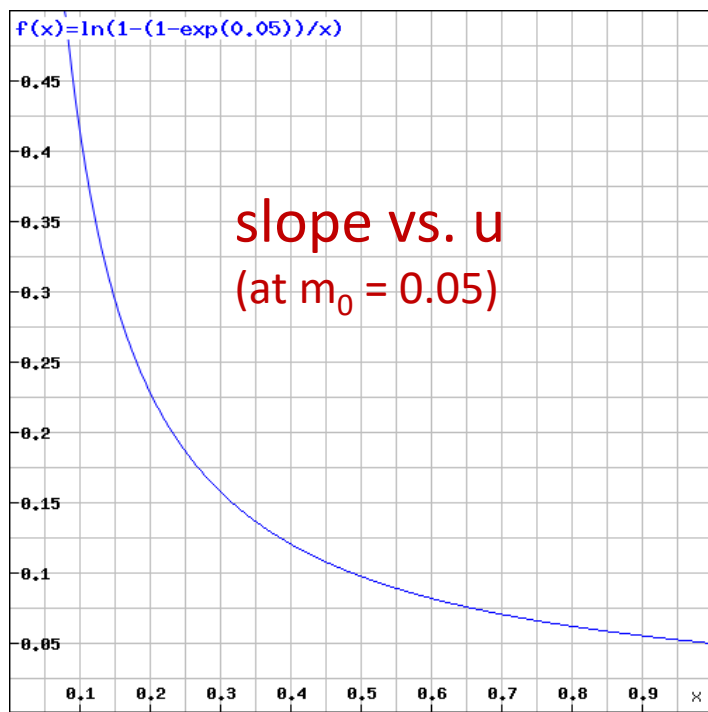
Realistically our wavelengths now are  $> 1 \text{ cm}$ , can't "rearrange" friction media

# Coil training with lower friction

$$f(n) \equiv \ln\left(1 - \frac{I_{n+1}}{I_{ss}}\right) = \ln\left(1 - \frac{I_1}{I_{ss}}\right) - n \underbrace{\ln(1 + \beta/\alpha)}_{|\text{slope}|}$$

$\alpha$  is proportional to the static friction coefficient  $\mu_s$ ; let's denote by  $u\alpha$  any change in the parameter caused by change in static friction ( $u\alpha \sim u\mu_s$ ) and the initial/nominal slope at  $u=1$  by  $m_0$ .

$$\text{slope} \equiv m = \ln\left(1 - \left(\frac{1-e^{m_0}}{u}\right)\right)$$



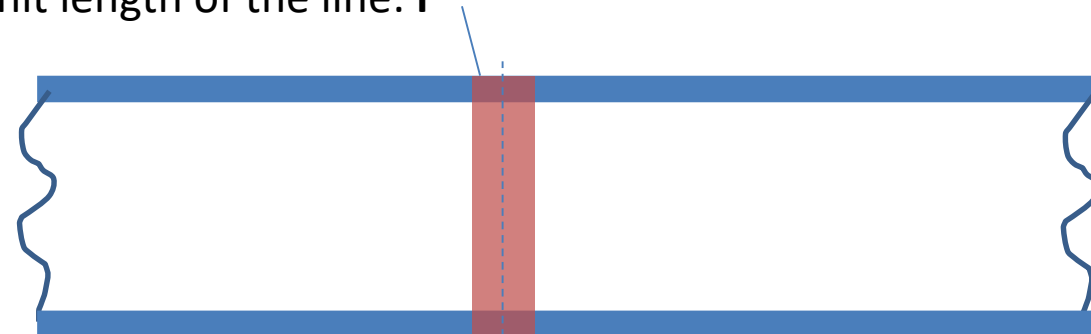
<https://rechneronline.de/function-graphs/>

# Vibrations in magnetic field

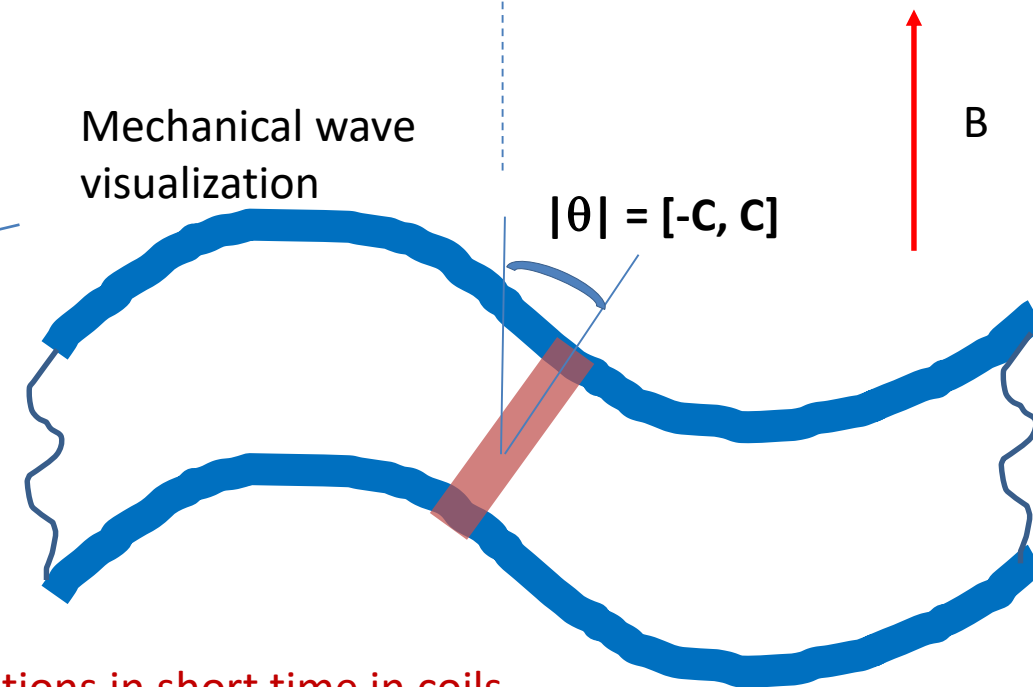
- If we have a non-superconducting closed-circuit area with varying magnetic flux density, we'll have energy losses
- One way to have that in a coil is by vibrations though they have to have specific characteristics to fulfil the conditions above
- The power losses need to be compensated by the vibration energy source

This is not the only way to describe the process, the important part is that vibrations of the proper kind will force repeatable flux changes through a surface.

Unit length of the line:  $l$



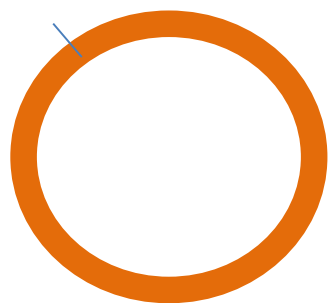
Mechanical wave visualization



Now, I want to check if it is possible at all to have sizable energy depositions in short time in coils.

# Vibrations in magnet cables

Copper line cross-section



Let's take two scales: strand and filament

**Strand:**

diameter  $d = 2r \sim 1 \text{ mm}$   
average copper tube thickness  
 $a \sim 0.07 \text{ mm}$

**Filament:**

$r = 25 \times 10^{-6} \text{ m}$  (Cu matrix cell)  
 $a = 3 \times 10^{-6} \text{ m}$

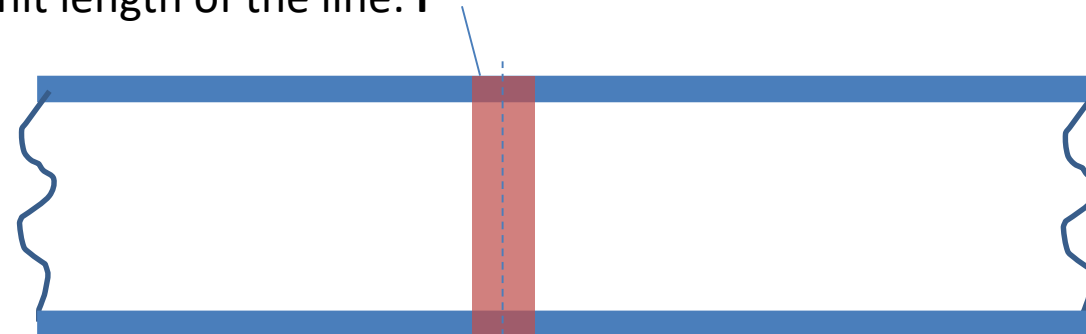
Axis of rotation  
(vibration) with  
frequency  $\omega$   
( $\sim 2\pi \cdot 20 \text{ kHz}$ )

$$\theta = C \sin(\omega t)$$

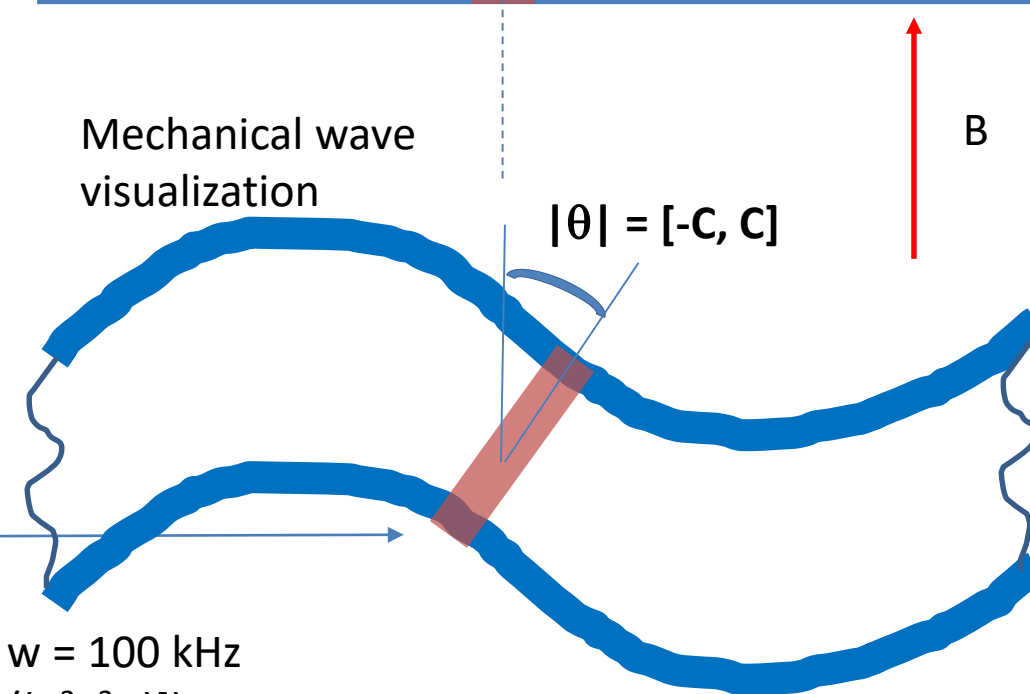
$B \sim 10 \text{ T}$   
magnetic flux density

If the amplitude  $A$  of vibrations is 10 microns and  $\omega = 100 \text{ kHz}$   
then  $C \sim 150 \mu\text{rad} \sim \pi/20000$  ( $\sim A/(\lambda/4) = \sqrt{8P/(\pi^2 v^3 \mu)}$ )

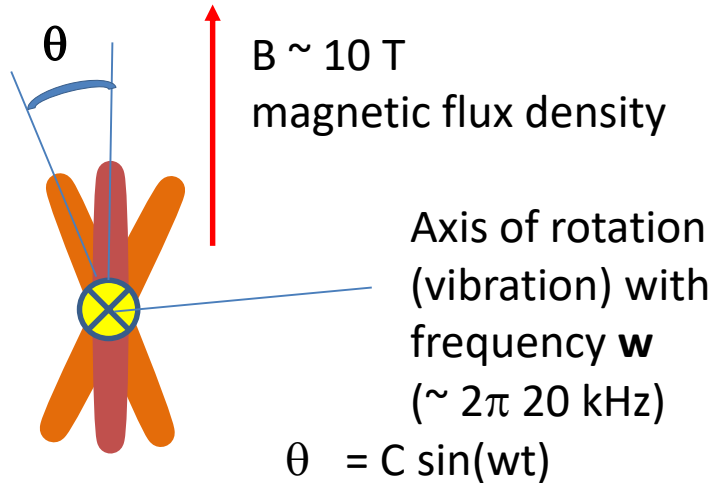
Unit length of the line:  $l$



Mechanical wave visualization



# Power losses



magnetic flux through this ring-like area

$$\Phi = B \pi r^2 \sin(\theta) = B \pi r^2 \sin(C \sin(wt))$$

induced emf (electro-motive force) due to those vibrations

$$\varepsilon = \partial\Phi/\partial t = B \pi r^2 w \cos(\theta) \text{sqrt}[C^2 - \theta^2]$$

Ignoring signs,  
self-inductance

resistance of the ring

$$R = \rho 2\pi r / (la) \equiv 2\pi r / (la \sigma)$$

Copper  
resistivity

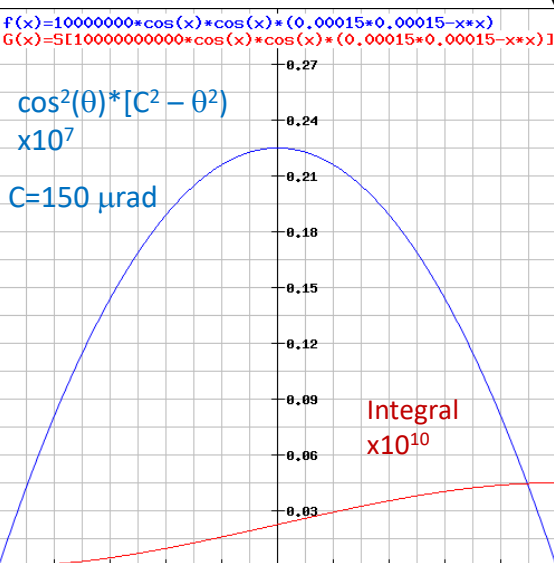
Copper  
conductivity

energy per unit time (and per unit I):

$$E/t = P = \text{emf}^2/R = \frac{1}{2} \sigma \pi l a r^3 B^2 w^2 \cos^2(\theta) [C^2 - \theta^2]$$

Over one period:  $\langle \text{ang} \rangle \equiv \langle \cos^2(\theta) * [C^2 - \theta^2] \rangle|_{C=150 \mu\text{rad}} = 15 \times 10^{-9}$  and  $l = \lambda = 2\pi v/w$

$$\langle \cos^2(\theta) * [C^2 - \theta^2] \rangle|_{C=10 \mu\text{rad}} = 14 \times 10^{-15}$$



<https://rechneronline.de/function-graphs/>



# Power dissipation at different scales

$$\langle P \rangle = \pi^2 \sigma a v \langle \text{ang} \rangle B^2 w r^3 \quad (\text{average power})$$

$$\sigma = 1 \times 10^{10} \text{ S/m at 2 K (RRR} \sim 150) \quad B = 10 \text{ T} \quad v = 4 \times 10^3 \text{ m/s} \quad w = 10^5 \text{ Hz} \quad (\lambda \sim 25 \text{ cm})$$

strand

$$r = 0.5 \times 10^{-3} \text{ m}$$

$$a = 7 \times 10^{-5} \text{ m}$$

filament

$$r = 25 \times 10^{-6} \text{ m}$$

$$a = 3 \times 10^{-6} \text{ m}$$

$$a_s \times r_s^3 / (a_f \times r_f^3) = 1.7 \times 10^{11}$$

strand

$$\begin{aligned} \langle P \rangle &= \pi^2 \sigma a v \langle \text{ang} \rangle B^2 w r^3 \sim \\ &\sim 9.5 \times 10^{10} \times 7 \times 10^{-5} \times 4 \times 10^3 \times 15 \times 10^{-9} \times 10^2 \times 10^5 \times 125 \times 10^{-6} \sim \\ &\sim 5 \times 10^5 \text{ W} \end{aligned}$$

One can vibrate strands in that manner but at much lower amplitude  
(with  $C=10 \mu\text{rad}$ , which is  $A \sim 0.7 \mu\text{m}$ , it goes 6 orders of magnitude down)

filament

$$\begin{aligned} \langle P \rangle &= \pi^2 \sigma a v \langle \text{ang} \rangle B^2 w r^3 \sim \\ &\sim 9.5 \times 10^{10} \times 3 \times 10^{-6} \times 4 \times 10^3 \times 15 \times 10^{-9} \times 10^2 \times 10^5 \times 15.6 \times 10^{-15} \sim \\ &\sim 3 \times 10^{-6} \text{ W} \end{aligned}$$

then  $\sim 150$  filaments (or like-sized closed-circuit structures) per 40 strands don't need much power to vibrate with the assumed amplitude: total is  $\sim 20 \text{ mW}$  (ignoring any other constraining force).

(those are over  $1 \lambda$ )

Also interesting to consider – effect of emf induced in the superconductor on transport properties.

# “Optimal” power dissipation

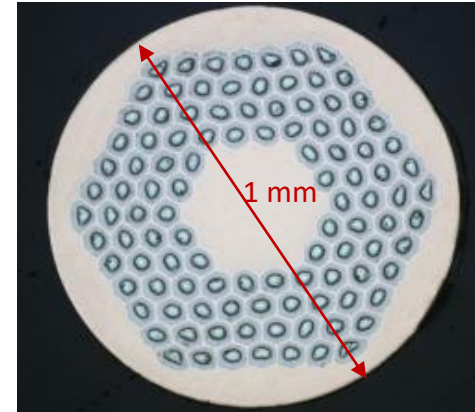
Looking at what energy can reasonably be dissipated in a coil (no other losses).

/Let's clarify again,  $\langle P \rangle$  is for one period, integrated over one wavelength. /

At  $w=10^5$ , one wavelength is  $\sim 25$  cm,  $T \sim 60 \mu s$ ; a short coil is  $\sim 125$  m, so  $500 \lambda$ .

If we have **0.25 kW discharged in 30 ms** this is  $0.5 \text{ J}/60 \mu s$  or  $0.5 \text{ J}$  per  $1 \lambda$ . Having 40 strands:  $P_{\text{strand}}$  is  $12.5 \text{ mJ}/1 \lambda$  (or **0.5 mJ per cm of strand**)

Finding a hypothetical “tube” radius where 1/10 of the power (2 mJ per cm of cable in 30 ms) would be dissipated:



108/127 RRP wire

$$r^3 = \langle P \rangle (\pi^2 \sigma a v \langle \omega \rangle B^2 w)^{-1}$$

Keeping all parameters the same, including  $a = 3 \times 10^{-6} \text{ m}$  (filament parameter)

$$\begin{aligned} r &= \text{cubic root} \left( P \times (\pi^2 \sigma a v \langle \omega \rangle B^2 w)^{-1} \right) \sim \\ &\sim \text{cubic root} (1.25 \times 10^{-3} \times 9.5^{-1} \times 10^{-10} \times 3^{-1} \times 10^6 \times 4^{-1} \times 10^{-3} \times 15^{-1} \times 10^9 \times 10^{-2} \times 10^{-5}) \sim \\ &\sim 0.2 \times 10^{-3} \text{ m (this is } 2/5 \text{ the strand radius)} \end{aligned}$$

How filaments or parts of strands could move/vibrate within the strand is hard to model (for me).

However, we can expect significant energy loss **IF** we manage to supply proper wave excitations wrt magnetic field lines.

# Summary 2

- Vibrations in coils (conductor or else) has the potential to affect training performance
  - There is no direct studies on coils yet
  - Projected wave parameters are in an accessible range
  - We don't have proper expertise but on the other hand there is no necessarily fine tuning of parameters
- Vibrations also have the potential to induce energy loss in magnet conductor in magnetic field
  - Wave parameters are in an accessible range
  - Generally, this requires much more demanding wave parameters tuning
  - On the other hand, lack of additional energy losses benefits friction-reduction

# Possible path forward

- **Dedicated** research by focused individual(s)
  - Postdoc (or may be a PhD student)
  - Development of simulations/software framework
  - Dedicated tests at small-scale (benchmarking)
  - Larger scale test plans
- Dedicated material support
  - Specialized ultrasonic machine(s)
  - Likely close coordination with manufacturers
  - Integration with other developments

There are no projections yet any of those will be supported. Then we can invest in small auxiliary experiments, extending run plans in other tests. We could hope that those small-scale tests will give us more insights or results making attraction of funds more likely.

There are no projections those will get enough “critical mass” either. So “we” can decide to work along those lines or dedicated individuals may have to decide how dedicated they can afford to be.

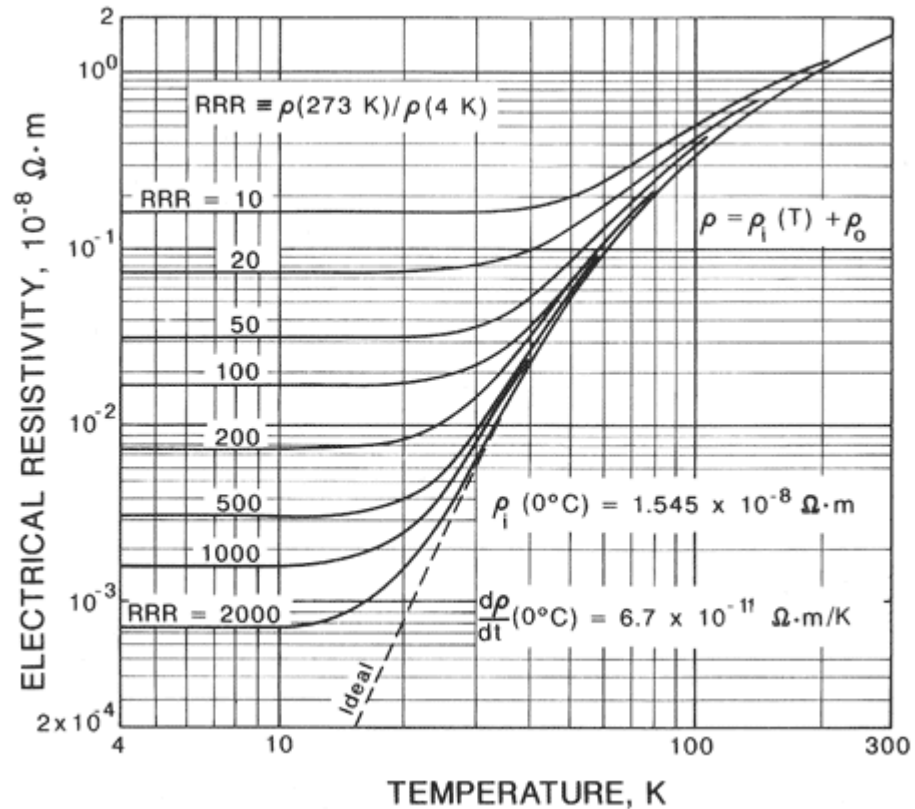
## US Magnet Development Program (MDP) Goals:

### GOAL 1:

Explore the performance limits of Nb<sub>3</sub>Sn accelerator magnets with a focus on minimizing the required operating margin and significantly reducing or eliminating training.

*What resources and distribution we want to provide for reaching Goal 1?*

# Spare



Note that for rotating frame in magnetic field and using the same formula so far

$$\langle \cos^2(\theta) * [C^2 - \theta^2] \rangle |_{C=\pi} \approx 1/2$$

Thus

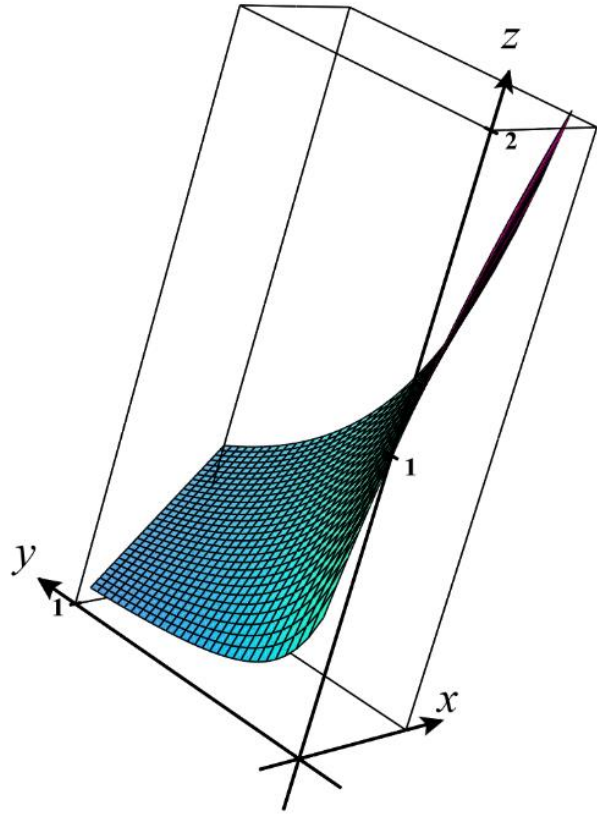
$$\langle \varepsilon^2 \rangle = \frac{1}{2} (B \pi r^2 w)^2$$

which is the known solution

<https://www.copper.org/resources/properties/cryogenic/images/Electrical-Resistivity.gif>



# Spare



$$z = \ln(1 - (1 - \exp(x))/y)$$

<https://www.monroecc.edu/faculty/pauseeburger/calcnsc/CalcPlot3D/>